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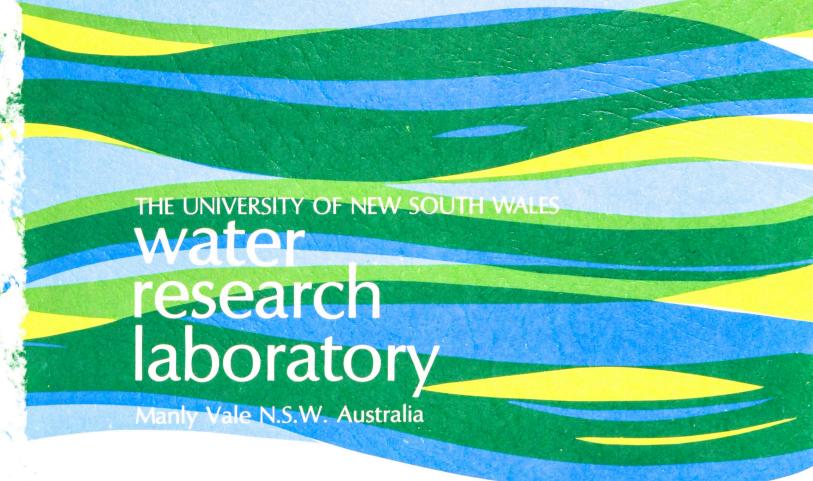
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# **ANALYSIS OF PIPE TESTS**

by

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T.R. Fietz, and K.B. Higgs

Research Report No. 174 August 1989

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# THE UNIVERSITY OF NEW SOUTH WALES

WATER RESEARCH LABORATORY

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# ABSTRACT

Analysis of pipe tests to determine friction coefficients and their uncertainty is discussed. Optimisation methods are applied to finding the pipe diameter, friction coefficient, and fitting loss coefficient from a pipe test. A FORTRAN program implementing these methods is included.

# NOTATION

a	Coefficient for least squares regression line
b	Slope for least squares regression line
$C_o$ to $C_5$	Coefficients
d	Diameter
d <sub>g</sub>	Global d
d <sub>h</sub>	Upper limit of d range for search
d <sub>high</sub>	Upper d value
dı	Lower limit of d range for search
$d_{low}$	Lower d value
e <sub>d</sub>	Uncertainty in d
e <sub>k</sub>	Uncertainty in k
e <sub>l</sub>	Uncertainty in l
e <sub>Q</sub>	Uncertainty in Q
es	Uncertainty in S
e <sub>t</sub>	Uncertainty in t
e <sub>μ</sub>	Uncertainty in $\mu$
e <sub>v</sub>	Uncertainty in v
e <sub>ρ</sub>	Uncertainty in p
f	Darcy friction factor
f(x)	Function of x
f(x <sub>*</sub> )	Minimum of f(x)
$f^1(x)$	First derivative of f(x)
g	Gravitational acceleration
h	Piezometric head
$\underline{\mathbf{h}}_{\mathbf{f}}$	Drop in HGL due to friction (or surface resistance) head loss
h	Mean of h observations
h <sub>o</sub>	h on best fit HGL
H <sub>f</sub>	Friction (or surface resistance) head loss
$H_L$	Total head loss
H <sub>o</sub>	Observed total head
$H_v$	Fitting (or minor) head loss
HGL	Hydraulic grade line (piezometric head line)
I <sub>d</sub>	Number of iterations for d search
-a I <sub>k</sub>	Number of iterations for k search
I <sub>KL</sub>	Number of iterations for $K_L$ search
k	Equivalent sand grain roughness
	Global k
k <sub>g</sub>	

k <sub>h</sub>	Upper limit of k range for search
k <sub>l</sub>	Lower limit of k range for search
k <sub>max</sub>	k at upper limit of uncertainty
k <sub>mean</sub>	k from mean properties
k <sub>min</sub>	k at lower limit of uncertainty
K <sub>L</sub>	Fitting loss coefficient
K <sub>Lg</sub>	Global K <sub>L</sub>
$K_{L_h}$	Upper limit of $K_L$ range for search
K <sub>L1</sub>	Lower limit of $K_L$ range for search
l	Distance along pipe
$\frac{l}{l}$ .	Mean of <i>l</i> observations
L	Pipe length
n	Number of sets of pipe test observations
Ν	Number of iterations
р	Static pressure
Q	Discharge
Qo	Observed discharge
$Q_{high}$	Upper Q value
Q <sub>low</sub>	Lower Q value
r	Golden section ratio
R	Residual head loss or discharge
R	Reynolds number
S	Energy gradient (downward +ve)
S <sub>high</sub>	Upper S value
S <sub>low</sub>	Lower S value
S <sub>mean</sub>	Mean S
S <sub>*</sub>	Published S value
t	Temperature
tp	Student's t coefficient
v	Pipe mean velocity
x	Abscissa, independent variable
X*	x at function minimum
<b>x</b> <sub>1</sub>	Lower limit of x range containing minimum
x <sub>2</sub>	Upper limit of x range containing minimum
N7 N7	Intermediate x values between $x_1$ and $x_2$
x <sub>3</sub> , x <sub>4</sub>	
х <sub>3</sub> , х <sub>4</sub> У	Ordinate, dependent variable

γ	Unit weight of water
δx	x interval
Δd	d tolerance
$\Delta h_{max}$	Maximum perturbation about best fit HGL
Δk	Small k increment
$\Delta l$	Spacing of head tapping points
ΔS	Small S increment
$\Delta t$	Temperature increment
$\Delta \mathbf{x}$	Range of uncertainty in $x_*$
$\theta_d$	Sensitivity coefficient for d
$\theta_Q$	Sensitivity coefficient for Q
$\theta_{S}$	Sensitivity coefficient for S
$\theta_{v}$	Sensitivity coefficient for $v$
μ	Dynamic viscosity of water
ν	Kinematic viscosity of water
ν <sub>o</sub>	Observed kinematic viscosity
$v_{high}$	Upper v value
$v_{low}$	Lower v value
ρ	Density of water

#### INTRODUCTION

Computation of total head loss due to friction (or surface resistance) and minor (or fitting) loss is necessary for the hydraulic design of water supply pipelines and networks.

The reliability of the head loss estimates is affected by the uncertainty in the design values selected, in particular the value characterising the pipe material for the calculation of friction loss.

Design values for computing friction loss and minor loss are derived from pipe tests by fitting assumed mathematical relations to the test data. In this report the Darcy-Weisbach equation is taken as the relation for friction loss, in conjunction with the Colebrook-White equation for finding the equivalent sand grain roughness, which is the design value specifying the pipe wall material. The assumed mathematical relation for minor loss uses a constant minor loss coefficient applied to the pipe velocity head.

This report aims to:

- (i) Develop statistically proper, programmable methods for determining the equivalent sand grain roughness and its uncertainty from a pipe test where the pipe diameter is known.
- (ii) To show that the least squares minimisation method may be used for analysis of pipe test data where various combinations of pipe diameter, equivalent sand grain roughness and minor loss coefficient are required.

A FORTRAN computer program implementing the proposed methods is included, together with the results of applying the program to both synthetically generated and published pipe test data.

# **BASIC EQUATIONS**

Figure 1 shows the head loss components for steady flow of water under pressure in a circular pipe. The total head loss  $H_L$  may be expressed as:

$$H_{L} = H_{f} + H_{v} \tag{1}$$

where  $H_f$  is the friction loss and  $H_v$  is the fitting loss, assumed concentrated at the end of the pipe length 1 being considered.

The fitting loss is given by:

$$H_{v} = K_{L} \frac{V^{2}}{2g}$$
<sup>(2)</sup>

where  $K_L$  is the fitting "loss coefficient" and V the pipe mean velocity.  $K_L$  depends mainly on fitting shape, and to a lesser extent on pipe Reynolds Number, pipe roughness, proximity to other fittings, etc.  $K_L$  may be determined accurately as a function of Reynolds Number and other factors [31], but for water supply calculations  $K_L$  is usually taken as a constant value for a particular fitting shape [2].

For a prismatic pipe the friction loss  $H_f$  is equal to the drop in the Hydraulic Grade Line  $h_f$ , given by the Darcy-Weisbach equation:

$$h_f = f \frac{L}{d} \frac{V^2}{2g}$$
(3)

The Darcy friction factor f depends on the Reynolds Number **R** and the pipe relative roughness  $\frac{k}{d}$ , where k is the "equivalent sand grain roughness", a linear measure of the roughness of the pipe wall. The most popular expression relating f to **R** and  $\frac{k}{d}$  is the Colebrook-White equation [14]:

$$\frac{1.0}{\sqrt{f}} = -2\log_{10}\left(\frac{k}{3.7 \,d} + \frac{2.51}{R\sqrt{f}}\right)$$
(4)

The Colebrook-White equation works well for small (less than 300 mm dia.) water supply pipes, both new and aged. For larger conduits deviations from the Colebrook-White function are frequently reported, for example from tests on concrete pipes [38], concrete tunnels [18], coated steel pipes [6], [9], [30], steel lined tunnels [18], and unlined rock tunnels at low flow rates [34]. Barr [3], [4], has derived formulae to cover these deviations but the expressions require six coefficients while Equation (4) requires only the equivalent sand grain roughness k.

For discussion of pipe test results the Moody Chart [32] provides the most convenient graphical representation of Equation (4).

Equation (4) is implicit in the friction factor f, but may be readily solved iteratively by "successive substitution" [43]:

$$\frac{1.0}{\sqrt{f_{i+1}}} = -2\log_{10}\left[\frac{k}{3.7 \,\mathrm{d}} + \frac{2.51}{\mathbb{R}\sqrt{f_{i}}}\right]$$
(5)

(6)

where  $f_i$ ,  $f_{i+1}$  are f values for iterations i, i+1 respectively. To commence iterations  $f_1$  may be taken as 1.0 or as f for a rough pipe, obtained by eliminating the  $\frac{2.51}{\mathbb{R}\sqrt{f_i}}$  term in Equation (5).

Putting V =  $\frac{4Q}{\pi d^2}$  in Equations (2) and (3), Equation (1) becomes:  $H_L = \left[ f \frac{L}{d} + K_L \right] \frac{8Q^2}{\pi^2 g d^4}$ 

When the pipe flow is wholly rough wall turbulent (on the far right of the Moody chart, Figure 4) the friction factor f takes on a constant value. The minor loss coefficient  $K_L$  is also constant so some ambiguity in f and  $K_L$  values might be expected when Equation (6) is used in procedures to analyse test results from a pipe operating as wholly rough wall turbulent flow and with minor losses.

Putting  $V = \frac{4Q}{\pi d^2}$  in the Reynolds Number  $\mathbb{R} = \frac{Vd}{v}$  makes the friction factor f in Equations (4) and (5) a function of (Q, k, d, v), where v is the kinematic viscosity of water.

Equation (6) may be re-arranged to make Q the dependent variable:

$$Q = \left\{ \frac{H_{L} \pi^{2} g d^{4}}{8 (f \frac{L}{d} + K_{L})} \right\}^{0.5}$$
(7)

When the pipe flow is smooth wall turbulent, or transitional from smooth wall turbulent to wholly rough wall turbulent, the friction factor f in Equation (7) is a function of Q, so Equation (7) is implicit in Q and requires an iterative solution.

Tests on a pipe of known length L yield n sets of observations of discharge  $Q_o$ , head loss  $H_o$ , and water temperature, from which the kinematic viscosity  $v_o$  may be found from tables. Equations (6) and (7) represent the assumed mathematical relations to be fitted to the test observations. A least squares minimisation procedure may be used to find the required pipe properties [10].

If Equation (6) is used the predicted head loss  $H_L$  ( $Q_o$ ) is found by using the observed discharge  $Q_o$  in Equation (6). The residual head loss R is the difference between the predicted head loss and the observed head loss  $H_o$ . An adequate estimate of the pipe properties is assumed to occur when the sum of the squares of the residuals  $\Sigma R^2$  is a minimum for all n sets of test observations.

For minimisation in terms of head loss the function to be minimised is therefore:

$$F(k, d, K_L) = \sum_{i=1}^{n} \left\{ H_{o_i} - H_L(Q_{o_i}) \right\}^2 = \sum_{i=1}^{n} R_i^2$$
(8)

In Equation (8) F (k, d,  $K_L$ ) indicates that one or more of k, d and  $K_L$  are unknown values. The term  $H_L(Q_{o_i})$  is the head loss predicted by using  $Q_{o_i}$  in Equation (6), and  $R_i$  is the residual head loss.

Minimisation in terms of head loss residuals is preferable to minimisation in terms of discharge residuals [Equation (9) below] for two reasons:

- (i) The least squares fitting procedure assumes that the dependent variable varies randomly about the fitted curve while the independent variable is known with negligible error [10]. As will be shown subsequently, the uncertainty in the head loss observation  $H_o$  is greater than the uncertainty in the discharge measurement  $Q_o$ , so  $H_o$  should be taken as the dependent variable.
- (ii) The iterative solution required to find the predicted discharge  $Q(H_o)$  from Equation (7), for use in Equation (9), is avoided.

Alternatively Equation (7) may be used for minimisation in terms of discharge where the function to be minimised is:

$$F(k, d, K_L) = \sum_{i=1}^{n} \left\{ Q_{o_i} - Q(H_{o_i}) \right\}^2 = \sum_{i=1}^{n} R_i^2$$
(9)

Again F(k, d,  $K_L$ ) indicates that one or more of k, d, and  $K_L$  are unknown. The term Q(H<sub>oi</sub>) is the discharge predicted by using H<sub>oi</sub> in Equation (7) and R<sub>i</sub> is the residual discharge.

One method of minimisation requires the partial derivatives  $\frac{\partial F}{\partial k}$  and  $\frac{\partial F}{\partial d}$  for Equation (9) with the loss coefficient  $K_L = 0$ . The derivatives are:

$$\frac{\partial F}{\partial k} = \sum_{i=1}^{n} 2 \left\{ Q_{o_i} + C_4 C_5 \right\} \frac{C_4}{3.7d}$$
(10)  

$$\frac{\partial F}{\partial d} = \sum_{i=1}^{n} 2 \left\{ Q_{o_i} + C_4 C_5 \right\} \left\{ \frac{-C_1}{C_3} \left[ \frac{kd}{3.7} + 1.5 C_2 \right] + 2.5 C_1 C_5 d^{1.5} \right\}$$
(11)  
where  $C_o = \left[ 2g \frac{H_{o_i}}{L} \right]^{0.5}$ ,  $C_1 = (\log_{10}e) \frac{\pi}{2} C_o$ ,  $C_2 = \frac{2.51v}{C_o}$   
and  $C_3 = \frac{k}{3.7d} + \frac{C_1}{d^{1.5}}$ ,  $C_4 = C_1 d^{2.5}$ ,  $C_5 = \log_e(C_3)$ 

# DETERMINATION OF DIAMETER d, EQUIVALENT SAND GRAIN ROUGHNESS k, AND LOSS COEFFICIENT K<sub>L</sub> FROM PIPE TESTS

# Case 1: Given d, to Find k

This is the most common case for analysis of both laboratory and field pipe tests. Any minor losses due to joints, or slight changes in alignment in installed pipes, are absorbed in selecting the k value so  $K_L$  is taken as zero.

Minimisation in terms of head loss residuals has been adopted for the reasons discussed previously. In Equation (8) k is the unknown and the required value  $k_g$  (global k) may be found by searching for the minimum of  $\sum R_i^2$  between stated limits  $k_l$  and  $k_h$ , using one of the minimisation procedures described in Appendix A. The search procedure is shown schematically in Figure 2(A).

Note that only one set of observations of  $Q_o$ ,  $H_o$ , and  $v_o$  is required to calculate f and **R** and to then select  $\frac{k}{d}$  from the Moody Chart. Similarly only one set of observations is required for the least squares method outlined above, although as many sets as possible should be used to improve

the reliability of the k estimate.

# Case 2: Given d and k, to Find K,

This case is applicable to a pipeline of known d and k which has appreciable minor losses generated by non-prismatic components such as bends, partially closed valves, etc. The procedure finds a single  $K_L$  value applicable over the range of test discharges

A similar least squares method to that used for Case 1 is employed. The unknown variable in Equation (8) is now  $K_L$ .

The search procedure is shown schematically in Figure 2(B) where the search limits are  $K_{L_1}$  to  $K_{L_b}$  and the global minimum is  $K_{L_e}$ .

Only one set of observations of  $Q_o$ ,  $H_o$  and  $v_o$  is required to find  $K_{L_g}$  but as many sets as possible should be used to assess the suitability of Equation (6) as a model for the  $H_L - Q$  relation for the pipe.

# Case 3: Given d, to Find k and K,

This case might be applied to test results from a pipeline of known d but where k has changed with use and appreciable minor losses are suspected.

Here the unknown variables in Equation (8) are k and  $K_L$  so the minimum of a two-variable function is required. As shown in Figure 3,  $\sum R_i^2$  as a function of k and  $K_L$  may be visualised as a three dimensional surface. The global minimum with values  $k_g$  and  $K_{L_g}$  occurs at the lowest point of the valley line.

Numerous techniques for optimising (i.e. finding the global minimum) of a multivariate function are known [25]. For the present report it has been decided to use a crude but readily visualised and easily programmed parametric search technique. The technique involves recursive application of a procedure for minimisation of a one-variable function and was first applied to analysing pipe test results by Stuckey [39].

The search procedure is shown schematically in Figure 2(C). The search is made between outer limits  $k_1$  and  $k_h$ . For each k value an inner search is made in the normal direction between limits  $K_{L_1}$  and  $K_{L_h}$  to locate the local minimum  $K_L$ . At the global minimum the values  $k_g$  and  $K_{L_g}$  locate the global minimum value of  $\sum R_i^2$ .

If  $I_k$  is the number of iterations required to search in the k direction, and  $I_{K_L}$  to search in the K<sub>L</sub> direction, then the total number of iterations is  $I_k \times I_{K_L}$ .

The minimum number of sets of test observations required to find k and  $K_L$  is n = 2.

# Case 4: Find d and k

This case might be applied to test results from an old inaccessible pipeline to obtain an "effective diameter d" and "an effective sand grain roughness k" which best fits the test results. The unknown variables in Equation (8) are d and k.

The search procedure is shown schematically in Figure 2 (D). The outer search is between limits  $k_1$  and  $k_h$  and the inner search between  $d_1$  and  $d_h$ .

Stuckey [39] solved this case using minimisation of the discharge residuals defined by Equation (9).

# Case 5: Find d, k and $K_{II}$

This case may be applied to test results from a pipeline with little information as to size, condition or presence of fittings. Case 5 may be applied as an alternative to Case 4. The sum of the squares of the residual head losses  $\sum R_i^2$  indicates which solution is a better fit to the test data.

This case requires searching at three levels, as shown schematically in Figure 2(E).

The procedure may be visualised as examining numerous three dimensional surfaces, each labelled with a  $K_L$  = constant value, to find the surface which has the smallest local minimum value of  $\sum R_i^2$ . The smallest local minimum becomes the global minimum with the required variable values  $K_{L_g}$ , d<sub>g</sub> and k<sub>g</sub>.

If the number of iterations required to search for  $K_L$ , d and k is  $I_{K_L}$ ,  $I_d$  and  $I_k$  respectively, then the total number of iterations to find the global minimum is  $I_{K_L} \times I_d \times I_k$ . The computer time required for a three-variable function makes this the practical limit of application of parametric searching methods for use on micro and mini computers.

The minimum number of sets of test observations required to find d, k and  $K_L$  is n=3.

# SELECTION OF A MINIMISATION PROCEDURE

Two procedures for minimisation of a one-variable function are described in Appendix A. These have been applied to the synthetically generated pipe tests described below with the following results:

#### Minimisation by Bisection of the First Derivative Function

This was tried for Case 4 problems, that is to find d and k, using Equation (9) as the function to be minimised, with  $K_L = 0$  for this case.

The outer search is between limits  $k_l$  and  $k_h$  with the object of making  $\frac{\partial F}{\partial k}$  from Equation (10) equal to zero by the method of bisection. For each k value during the outer search an inner search is made between  $d_l$  and  $d_h$  to make  $\frac{\partial F}{\partial d}$  equal to zero by bisection.

The method performed satisfactorily in that the global minimum values  $k_g$  and  $d_g$  were correctly predicted. From Equations (A1) and (A2) in Appendix A, the number of iterations for convergence by bisection is about 0.7 times the number for a golden section search. The additional computer time required to evaluate the partial derivatives  $\frac{\partial F}{\partial k}$  and  $\frac{\partial F}{\partial d}$  appears to outweigh the apparent saving in the lower number of iterations so minimisation by bisection was not pursued further.

#### Minimisation by Golden Section Search

This method was found to be reliable and to converge for Cases 1 to 5 of pipe test analysis, and has been adopted as the standard procedure.

For Cases 3, 4 and 5, requiring two or more recursive applications of the golden section search algorithm, individually labelled subroutines have been used in the computer program as recursion is not implemented in some versions of FORTRAN.

## **COMPUTER PROGRAM**

A computer program for solving pipe test Cases 1 to 5 is shown as Appendix C. The program is written in FORTRAN 77 and was used on a Digital MicroVAX II computer.

The program is developmental in that it was assembled using components from other programs to demonstrate the techniques described above. The program is not intended as an example of good or efficient programming.

# PROGRAM TESTING USING SYNTHETIC PIPE TEST RESULTS

The program was tested by application to synthetic pipe test results generated by using Equation (6) to find head losses for at least ten equally spaced discharges. No attempt has been made to apply a random component to the head loss  $H_o$  values generated, although this could be done using the method described for generating random piezometric head values in the "Uncertainty in the Energy Gradient S" section below. The pipes selected cover the range of operation for typical

water supply pipes [21]. Some of the data is plotted on a Moody Chart, shown as Figure 4. Some very rough pipes, which would plot above the maximum f value on Figure 4, were also tested.

The program predicted the unknowns for Cases 1 to 4 without fail. The only tests where any discrepancy occurred was in application of Case 5 to very rough pipes with minor losses approximately equal to friction losses. In these cases the predictions of d, k and K<sub>L</sub> values were improved by using discharge increments proportional to  $\log_{10} (Q)$ . The test discharges were calculated by dividing  $\left\{ \log_{10} (\max, Q) - \log_{10} (\min, Q) \right\}$  into the required number of increments and then taking the antilogs to find the discharges. This compresses the flowrates towards the lower end of the range. Some typical results are shown in Table 1 below:

	d (mm)	k (mm)	K <sub>L</sub>
Values used to generate data	245.9	10	40
Predicted values, linear Q increments	245.04	10.221	31.81
Predicted values, $log_{10}$ (Q) increments	246.6	9.826	46.59
Dine . Cast Iron Tuberculated High Mine			
Pipe: Cast Iron, Tuberculated, High Mine	r	k (mm)	K.
Pipe: Cast Iron, Tuberculated, High Mind	or Losses d (mm)	k (mm)	K <sub>L</sub>
Pipe: Cast Iron, Tuberculated, High Mino Values used to generate data	r	k (mm) 15	К <sub>L</sub> 100
	d (mm)		

TABLE 1:	Case 5,	Find d,	k and	K <sub>L</sub> ,	Very	Rough Pi	pes
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For all Case 5 applications shown in Table 1 the searches converged to give  $\sum R_i^2$  of zero, i.e. a head loss calculated using the predicted values of d, k and K<sub>L</sub> for a given discharge was equal to the test head loss for that discharge. The solution is therefore dependent on the discharge selection for the pipe tests. Minimisation using discharge as the working variable may improve the solution but this has not been tried.

# APPLICATION OF THE PROGRAM TO PUBLISHED PIPE TEST RESULTS

The criteria for selecting pipe test results from the literature were, firstly, an adequate number of sets of  $Q_o$ ,  $H_o$  and  $v_o$  observations, and secondly, preferably more than two head measuring tapping points along the line. Several tapping points are required to estimate the uncertainty in the slope of the hydraulic grade line, as discussed later. In selecting the sets of test data from a particular source some obvious "outliers" [31] were arbitrarily rejected.

Some of the pipe test data selected are shown on the Moody Chart, Figure 5 and the program test results are shown in Table 2. The tests labelled "mean properties", "properties for min. k", and "properties for max. k" are explained in the section on estimation of the uncertainty in k below.

Published k values were available for tests 5, 8 and 9, thus allowing comparison with k values from the Case 1 analyses. The published k values for tests 5 and 8 were obtained by the standard method of plotting  $(f, \mathbf{R})$  points on the Moody Chart and selecting by eye a  $\frac{k}{d}$  = constant line of best fit. The published k value for test 9 was obtained by comparing a logarithmic plot of observed  $(Q_o, H_o)$  values with that for new pipe. The known k value for the new pipe was increased to allow for the increased head loss in the aged pipe being tested. The Case 1 analyses of tests 5, 8 and 9 gave k values ranging from 30% greater to 40% smaller than the published k values. The range of uncertainty in k for tests 5 and 9 was estimated from Case 1 analyses of tests 6 and 7, and tests 10 and 11, respectively. For test 5 the k value was 0.055 mm and the range of uncertainty was from 0.038 mm to 0.077 mm, to 95% confidence level. For test 9 the k value was 0.0269 mm and the range of uncertainty from 0.0 mm to 0.060 mm.

The differences between published k values and those from the present method, and the wide range of uncertainty in the k values, suggests that a more critical view should be taken of published k values.

The Case 3 analyses are based on Equation (6), which allows for both friction loss and fitting loss whereas the Case 1 analyses are based on Equation (3), which allows for friction loss only. Comparing the residual errors  $\sum R_i^2$  shows which model is the better fit to the pipe test data. An obvious reduction in  $\sum R_i^2$  occurs for tests 1, 2 and 3, showing that the Case 3 method gives a better fit. Slight reductions in  $\sum R_i^2$  occur for tests 8 and 9 but these could be truncation errors so no firm conclusion can be drawn. Introduction of fitting loss had no effect on tests 4 and 5.

The Case 4 and Case 5 analyses provide the best test of the mathematical model, in particular in predicting the pipe diameter. A Case 4 analysis is based on Equation (3), for friction loss only, and estimates d and k for a set of pipe test observations  $Q_o$ ,  $H_o$  and  $v_o$ . For the present analysis d is known for all of the prismatic pipes tested. The Case 5 results gave d values within 5% of the known values. The residual errors  $\sum R_i^2$  are smaller than those from Case 1, indicating that the d and k values from Case 4 provide a better fit to the pipe test observations than the known d and the k given by the Case 1 analysis. Some improvement in the mathematical model appears to be possible, particularly in Equation (5), used to estimate the friction factor f. For an "unseen" pipe the values for d and k predicted by a Case 4 analysis are "effective d" and "effective k".

The Case 5 analyses are based on Equation (6), which includes friction and fitting losses, and estimates d, k and  $K_L$ . The estimated d values are within 6% of the known values, except for test 1 within 8%, and test 4, where d is overestimated by 32%. The effective d, k and  $K_L$  values found by a Case 5 analysis gave a lower residual error  $\sum R_i^2$  than that found using a Case 3 analysis for most of the tests examined.

# TABLE 2: RESULTS OF ANALYSIS OF PUBLISHED PIPE TESTS

					No.	No.	Published	l Values	C	Case 1		Case	3		Case 4			Ca	se 5	
Test No.	Source	Ref. No.	Pipe Details	Site	Data Sets	Head Taps	d(mm)	k(mm)	k(mm)	$\sum R_i^2$	k(mm)	KL	$\sum R_i^2$	d(mm)	k(mm)	∑R <sup>2</sup>	d(mm)	k(mm)	KL	$\sum R_i^2$
1	Heywood, 1925	26	4 in. Galv. iron All 30 data sets	Lab.	30	2	103.48		0.131	0.02289	0.016	0.57	0.01150	98.21	0.031	0.01184	111.77	0.0	1.53	0.01103
2	Heywood, 1925	26	4in. Galv. iron 14 sets known temp.	Lab.	14	2	103.48		0.131	0.02059	0.012	0.61	0.01072	97.62	0.025	0.01088	110.40	0.0	1.39	0.01054
3	Heywood, 1925	26	2in. Galv. iron 16 sets known temp.	Lab.	16	2	51.15		0.262	0.003075	0.10	1.3	0.00227	49.30	0.1262	0.002267	49.30	0.1262	0.0	0.002267
4	Burke, 1955	9	Steel penstock, hot enamel coat	Field	21	2	1289.0		0.060	0.159372	0.060	0.0	0.159373	1304.54	0.0892	0.158247	1698.0	0.0	3.07	0.15389
5	Levin, 1972	30	Machined casting, cold bitumen spray, mean properties.	Lab.	13	20	209.5	0.042	0.055	1.120746	0.055	0.0	1.120755	221.68	0.229	0.73428	221.68	0.229	0.0	0.73428
6	Levin, 1972	30	As above, min. k properties	Lab.	13	20	209.08		0.038	1.58576										
7	Levin, 1972	30	As above, max. k properties	Lab.	13	20	209.92		0.077	0.901908										
8	Dudgeon, 1983	17	Steel, cement mortar lined	Lab.	18	2	287	0.01	0.008	0.00661	0.0068	0.011	0.006603	286.86	0.007	0.0066033	289.66	0.004	0.223	0.006602
9	Foster, 1968	23	Asbestos cement, mean properties	Field	9	4	363.22	0.046	0.0269	0.0001916	0.0248	0.046	0.0001914	362.79	0.0236	0.0001909	362.79	0.0236	0.0	0.0001909
10	Foster, 1968	23	As above, min. k properties	Field	9	4	362.48		0	0.00136										
11	Foster, 1968	23	As above, max. k properties	Field	9	4	363.96		0.060	0.00127										

# **ESTIMATION OF THE UNCERTAINTY IN THE EQUIVALENT SAND GRAIN ROUGHNESS & FROM CASE 1 ANALYSES**

Levin [30] and Dudgeon [17] have combined the uncertainties of the other variables in Equation (3) to estimate the uncertainty in the friction factor f. This uncertainty in f only applies to the particular pipe tested whereas an uncertainty in k can be applied to any diameter pipe made of the same material.

Replacing  $\log_{10}$  by  $\log_{e}$  and using  $\mathbf{R} = \frac{4Q}{\pi dv}$  in Equation (4) gives:

$$\frac{1.0}{-2C_1 \sqrt{f}} = \log_e \left( \frac{k}{3.7d} + \frac{2.51 \pi dv}{4Q \sqrt{f}} \right)$$
(12)

where  $C_1 = \log_{10} (e)$ .

Taking antilogs and re-arranging gives an expression for k:

$$k = 3.7d \left[ e^{\frac{1.0}{-2C_1\sqrt{f}}} - \frac{2.51\pi d\nu}{4Q\sqrt{f}} \right]$$
(13)

Using the energy gradient  $S = \frac{h_f}{L}$  and  $V = \frac{4Q}{\pi d^2}$  in Equation (3) gives f for use in

Equation (13):

$$f = \frac{g\pi^2 d^5 S}{8Q^2}$$
(14)

The problem is to incorporate the uncertainties in the independent variables, S, D, Q and v to find the uncertainty in the dependent variable k defined by Equation (13). Two methods have been tried, direct combination and statistical combination.

#### **Direct Combination of Uncertainties**

If the mean value of S is  $S_{mean}$  then S may vary from  $S_{low} = S_{mean} - e_s$  to  $S_{high} = S_{mean} + e_s$ , where  $e_s$  is the uncertainty in S to some confidence level, say 95% [28]. Similarly d may vary from d<sub>low</sub> to d<sub>high</sub>, and so on. Possible combinations of S<sub>low</sub>, S<sub>high</sub>, d<sub>low</sub>, d<sub>high</sub> etc have been tried for data from several of the synthetic pipe tests described previously (with estimated uncertainties to find S<sub>low</sub>, S<sub>high</sub> etc) and for data from Levin [30] (with published uncertainties). The combinations found to give the k extremes were:

For minimum k:  $S_{low}$ ,  $d_{low}$ ,  $Q_{high}$ ,  $v_{high}$ 

```
For maximum \kappa: S_{high}, d_{high}, Q_{low}, v_{low}
```

When converted to equivalent f and  $\mathbb{R}$  values the data sets for minimum and maximum k plot as lines "parallel" to the mean properties data set. This is shown for the test data from Levin [30] on Figure 5.

Using  $S_{low}$ ,  $d_{low}$  etc, a Case 1 analysis is applied to find the estimate of the minimum k. Similarly for the maximum k estimate. This method has been used for test numbers 6, 7, 10, and 11 in Table 2.

## Statistical Combination of Uncertainties

Using the Simplified Method of combination of uncertainties [28], [35], the uncertainty in k due to uncertainties in the independent variables is:

$$\mathbf{e}_{\mathbf{k}} = \left\{ \left( \theta_{s} \, \mathbf{e}_{s} \right)^{2} + \left( \theta_{d} \, \mathbf{e}_{d} \right)^{2} + \left( \theta_{Q} \, \mathbf{e}_{Q} \right)^{2} + \left( \theta_{v} \, \mathbf{e}_{v} \right)^{2} \right\}^{0.5}$$
(15)

where  $\theta_s$  is the sensitivity coefficient for S, equal to  $\frac{\partial k}{\partial S}$  from Equations (13) and (14), and so on for the other variables. Finding the partial derivatives is tedious, so the partial derivatives have been approximated by finite increments [28], that is  $\theta_s \approx \frac{\Delta k}{\Delta S}$  where  $\Delta k$  is the change in k due to a small change  $\Delta S$  in S.

This method has been applied to results from a careful series of laboratory pipe tests by Levin [30]. A 1% change in variables was used to calculate approximate sensitivity coefficients  $\theta$  and values of uncertainties e were as published. Results for four sets of data are shown in Table 3 below:

# TABLE 3. Results of Statistical Combination of Uncertainties

Q Q <sub>max</sub>	$(\theta_s e_s)^2$	$(\theta_d e_d)^2$	$(\theta_Q e_Q)^2$	$(\theta_v e_v)^2$	e <sub>k</sub> (mm) from Eqn. (15)	θ <sub>s</sub> e <sub>s</sub> (mm)					
1.0	3.11 E-10	1.09 E-11	1.45 E-11	4.26 E-16	1.834 E-2	1.76 E-2					
0.7	2.59 E-10	1.17 E-11	1.76 E-11	1.024 E-15	1.698 E-2	1.61 E-2					
0.392	1.87 E-10	1.21 E-11	2.75 E-11	3.06 E-15	1.504 E-2	1.37 E-2					
0.073	1.39 E-10	6.6 E-11	1.88 E-10	6.94 E-13	1.98 E-2	1.17 E-2					

Test data from Levin [30] for laboratory tests on cold bitumen sprayed pipe

The results in Table 3 indicate that the energy gradient term makes the greatest contribution to  $e_k$ , except at very low discharges where the discharge term is of equal or greater importance. The pipe diameter term is insignificant, except at very low discharges, and the viscosity term can be ignored.

For the upper two-thirds of discharges, at least, taking the S term only in Equation (15) gives an approximate  $e_k = \theta_s e_s$ , which is slightly less than  $e_k$  found by using all terms.

Application of the statistical combination method at the design stage of a pipe test program should guide selection of measurement techniques and indicate areas where repetition of observations can reduce the uncertainty in k estimates.

The statistical combination method can only be applied to one set of test data at a time so some means of combining the n estimates of the uncertainty  $e_k$  values from n sets of data is required. On the other hand, the direct combination method above yields a "best fit" estimate of minimum k from the Case 1 analysis.

# Comparison of Uncertainties in k from the Direct Combination and the Statistical Combination Methods

The mean uncertainty in k from the direct combination method has been taken as:

$$\mathbf{e}_{\mathbf{k}} = 0.5 \left\{ \left\{ \mathbf{k}_{\max} - \mathbf{k}_{\min} \right\} \right\}$$
(16)

where  $k_{max}$  and  $k_{min}$  are the results of Case 1 analysis for data sets for minimum k and maximum k respectively.

The mean uncertainty for n data sets using the statistical combination method has been taken as the mean value  $\frac{1}{n} \sum_{i=1}^{n} e_{k_i}$ .

As expected [35], using the test data of Levin [30], the direct combination method gave a mean  $e_k$  about 10% greater than the mean  $e_k$  using statistical combination for four of the data sets.

The direct combination method is therefore slightly conservative in estimating the uncertainty in k but is recommended for its simplicity and ability to give "best fit" estimates of  $k_{min}$  and  $k_{max}$ .

# ESTIMATION OF UNCERTAINTIES IN THE INDEPENDENT VARIABLES

# S, d, Q and v

# **Measurement Methods for Pipe Tests**

A review of methods of measurement of pipe, flow and water properties is given in Appendix B. The uncertainties associated with measurement of these properties influence the uncertainty estimates for S, d, Q and v.

## Uncertainty in the Energy Gradient S

As shown in the section above, the uncertainty  $e_s$  in S probably has the predominant effect when finding the uncertainty  $e_k$  in k. Estimating S and its uncertainty  $e_s$  is usually the most difficult task in analysing pipe test results, particularly for large diameter smooth pipes, or smaller pipes at low discharges, where the head loss is small and S is very small.

For steady flow of an incompressible fluid in a prismatic pipe the energy line (and the parallel hydraulic grade line HGL) is theoretically linear, but even the most careful pipe tests produce a scatter of piezometric head values about a straight line. When tests are repeated at the same value of discharge the head observations at a particular cross-section show some random variation. Examples of piezometric head measurements for some laboratory and field tests are shown in Figures 6 and 7 respectively.

The deviations of the observed piezometric head measurements from a straight line of best fit are often too great to be explained by combining the uncertainties in the measurements used to determine the piezometric head h, particularly with laboratory tests, for example Figure 6(A). It is suggested that S, and its uncertainty  $e_s$ , be found by:

- (i) Adjusting the observations of the component variables used to find h to correct for any systematic errors, e.g. by calibration of pressure gauges.
- (ii) Then treating the HGL observations as a statistical problem of fitting a straight line where "x is known without error and y can vary". In the present case x is the distance l along the pipe and y is the piezometric head h. The uncertainty  $e_l$  in l is usually negligible compared to the uncertainty  $e_h$  in h.

A least squares linear regression provides a simple means of fitting a straight line to the n sets of (l,h) data [1], [10], [33], [35]. This method assumes that the h values are normally distributed about the regression line with constant variance.

The regression line is given by:

$$\mathbf{h} = \mathbf{a} + \mathbf{b}l \tag{17}$$

where 
$$b = \frac{\sum_{l=1}^{n} (l - \overline{l}) (h - \overline{h})}{\sum_{l=1}^{n} (l - \overline{l})^2}$$
 (18)

and 
$$a = h - bl$$
 (19)

In Equations (18) and (19)  $\bar{h}$  and  $\bar{l}$  are mean values, given by  $\bar{h} = \frac{1}{n} \sum_{1}^{n} h$  and  $\bar{l} = \frac{1}{n} \sum_{1}^{n} l$ , respectively. The energy gradient S is then equal to -b and the line of best fit passes through  $(\bar{l}, \bar{h})$ . S values found using this method are shown on some of the HGL lines in Figures 6 and

7, together with the published value  $S_*$ .

Assuming the estimates of b from repeated measurements follow Student's t distribution for a small number of observations n [10] [33], the uncertainty in b, to some prescribed confidence level, gives the uncertainty  $e_s$  in the energy gradient:

$$e_{s} = t_{p} \left\{ \frac{\sum_{l=1}^{n} (h-h_{o})^{2}}{\sum_{l=1}^{n} (l-\overline{l})^{2} (n-2)} \right\}^{0.5}$$
(20)

where  $t_p$  is a coefficient from Student's t distribution, n is the number of sets of (l,h) observations, and  $h_o$  is the h value computed from the line of best fit for a given l value.

The coefficients  $t_p$  may be obtained from tables [35] or from a graph [10].  $t_p$  values for confidence levels appropriate for pipe test analysis are shown graphically on Figure 8. A confidence level of 95% indicates that S is likely to be in the range  $(S-e_s)$  to  $(S+e_s)$  for 95% of the time.

In Figure 8 the abscissa is the number of degrees of freedom, equal to (n-2). At least three head tapping points are needed if an estimate of  $e_s$  is required. Figure 8 (B) (the linear plot) provides guidance in selecting the minimum number of head tapping points required to keep  $t_p$  (and hence  $e_s$ ) "reasonably low". For example, for a 95% confidence level six degrees of freedom, requiring eight tapping points, appears to be desirable.

Equation (20) shows that  $e_s$  is reduced by increasing the number n of head tapping points, which increases the value of the (n-2) term in denominator.

Equation (20) also indicates that  $e_s$  is reduced by increasing the spacing of the head tapping points. This effect has been investigated by applying Equation (20) to synthetic pipe test data. For given d, k, v and tapping spacing  $\Delta l$ , n sets of  $(l, h_o)$  points on the HGL are calculated. The  $h_o$  values are then perturbed by multiplying a perturbation  $\Delta h_{max}$  by a random number between -1.0 and +1.0. Results for two sets of data are shown in Figure 9. Increasing the tapping spacing  $\Delta l$  makes the energy gradient S approach the known value for the unperturbed HGL and decreases the uncertainty  $e_s$ .

Equation (20) suggests a possible measurement strategy might be to reduce  $e_s$  by increasing the  $(l-\overline{l})$  values, that is to concentrate the head tapping points at the upstream and at the downstream ends of the test section. There are two objections to this strategy:

(i) There would be no check on the assumed linearity of the HGL over the intermediate section of pipe. An obstruction causing localised minor loss could go undetected [23].

(ii) A statistical test would be required to show that the observations at each end were samples from the same population.

Investigation of any non-linearity of the HGL, and a check on the assumption of constant variance of the h values, would require repetition of the h observations for the same discharge, preferably taking observations over several cycles with Q increasing and Q decreasing [1]. This is usually impractical for field tests, which have to fit in with operating schedules, but could be performed for laboratory tests with little additional cost.

#### Uncertainty in Pipe Diameter d

Measurement of d volumetrically by filling a length of pipe with water should give  $\frac{e_d}{d} < 0.5\%$ , e.g. Levin [30] obtained  $\frac{e_d}{d} = 0.2\%$  for a 200 mm diameter pipe.

Measurements by precision mechanical instruments such as micrometers should give  $\frac{e_d}{d} < 0.5\%$ . Sufficient measurements should be taken to allow an elementary statistical analysis (see Appendix B).

When a d value is taken from manufacturers' catalogues the allowable tolerance in manufacture should be recognised. The allowable tolerance  $\frac{\Delta d}{d}$  may range from 4.5% for small (75 mm diameter) pipes to about 1.0% for larger (600 mm diameter) pipes [20].

# Uncertainty in Discharge Q

Orifice plates, venturis and flow nozzles manufactured, installed and operated according to standard codes should have  $\frac{e_Q}{Q} \le 1.25\%$  [22], [27], [28]. A pitot tube traverse, with appropriate corrections, should give  $\frac{e_Q}{Q} \le 1\%$  [7]. Electromagnetic flow meters are claimed to give  $\frac{e_Q}{Q} \le 1\%$  [30]. Laboratory volumetric tanks give  $\frac{e_Q}{Q}$  from 0.25% [38] to 0.5% [17], while volumetric gauging in natural reservoirs gives  $\frac{e_Q}{Q} \le 2\%$  [9]. Q measurement by stream gauging gives  $\frac{e_Q}{Q}$  about 6% [28].

# **Uncertainty in Water Properties**

Providing the water temperature can be found within  $\pm 0.5^{\circ}$ C, the uncertainty in density due to temperature variation can be neglected. The relative uncertainty in density  $\frac{e_{\rho}}{\rho}$  is about 0.15%, due to systematic uncertainty in published  $\rho$  values [22].

Dynamic viscosity  $\mu$  is much more sensitive to temperature change than density. A change in

temperature from 20 to 21°C will decrease  $\mu$  by about 2.5%. The uncertainty  $e_{\mu}$  for a temperature uncertainty  $e_t = \Delta t$  may be taken as the mean of the  $\mu$  deviation:

$$e_{\mu} = 0.5 \left[ \mu_{t-\Delta t} - \mu_{t+\Delta t} \right]$$
(21)

The percentage uncertainty  $\frac{e_{\nu}}{\nu}$  in the kinematic viscosity  $\nu = \frac{\mu}{\rho}$  may be taken as equal to  $\frac{e_{\mu}}{\mu}$  as  $\frac{e_{\rho}}{\rho} < 5 \frac{e_{\mu}}{\mu}$ , the criterion for ignoring the smaller value [28].

# CONCLUSIONS

- 1. It has been shown that least squares minimisation in terms of head loss residuals is a viable method for estimating the equivalent sand grain roughness k from a pipe test. The proposed method has a statistical basis and is preferable to the common method of drawing a line of best fit through plotted points on a Moody chart.
- 2. The uncertainty in the k value may be derived by direct combination of the estimated uncertainties in the pertinent variables. The uncertainty in the energy gradient S appears to be the major source of uncertainty in k. The uncertainty in S can be reduced by increasing the number of head tapping points and by increasing the spacing between the points.
- 3. Use of a total head loss relation (friction loss plus fitting loss) gave a better fit than friction loss only for some of the published pipe tests. This suggests that for some pipelines a  $K_L$  value, as well as a k value, should be specified.
- 4. The proposed least squares minimisation techniques have been shown to permit adequate estimation of "effective" diameter, k and  $K_L$  design values when various combinations of these values are unknown.



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## APPENDIX A: MINIMISATION OF A ONE-VARIABLE FUNCTION

Referring to Figure 10(A), the value of  $x_*$  is sought to define the single local minimum  $f(x_*)$  known to occur in the range  $x_1$  to  $x_2$ . The function f(x) may be "round bottomed" or "sharp bottomed" as shown.

(i) Minimum by Bisection of First Derivative Function [25]

If the first derivative function  $f^1(x)$  can be found then it crosses the x axis at  $x_*$ , as shown in Figure 10(B). The method of bisection (or interval halving) [10] may be used to obtain the root  $x_*$  of  $f^1(x)$ . The interval between  $x_1$  and  $x_2$  is bisected at  $x_3$  and the algebraic signs of  $f^1(x_1)$  and  $f^1(x_3)$ , and  $f^1(x_2)$  and  $f^1(x_3)$ , compared. The curve crosses the x axis when the signs are opposite, i.e. between  $x_1$  and  $x_3$  in Figure 10(B). The interval between  $x_1$  and  $x_3$  is then bisected and the procedure repeated until the required interval of uncertainty  $\Delta x$  is reached. Each iteration (after the first) requires only one evaluation of  $f^1(x)$  and the approximate number of iterations N required is:

$$N = \frac{\log_{e} \left[ (x_{1} - x_{2})/\Delta x \right]}{\log_{e} 2}$$
(A1)

## (ii) Minimum by Golden Section Search [13], [25]

This method is similar to the method of bisection in that the x interval containing the minimum is reduced by iteration. Each iteration reduces the interval from  $\delta x$  to  $r\delta x$ , where  $r=0.5(\sqrt{5}-1)$ , the golden section ratio, which satisfies the equation  $r^2 = 1-r$ . Referring to Figure 11(B), if  $x_*$  is known to be between  $x_1$  and  $x_2$ , then for the first iteration f(x) is evaluated at  $x_3$  and  $x_4$ . For the second and subsequent iterations only one determination of f(x) is required. The number of iterations N required to contain  $x_*$  within an interval of uncertainty  $\Delta x$  is:

$$N = \frac{-\log_{e} \left( (x_{2} - x_{1})/\Delta x \right)}{\log_{e} r}$$
(A2)

# **APPENDIX B: REVIEW OF MEASUREMENT METHODS FOR PIPE TESTS**

#### Measurement of Pipe Diameter d

The diameter of pipes small enough to be easily handled may be measured volumetrically by filling a length of pipe with water [26], [30], [37]. With very small pipes care should be taken to remove air bubbles adhering to the walls by wiping [37]. For a small rough pipe the diameter obtained by water filling may be less than that measured by sharp ended calipers or greater than that measured with flat ended calipers [37].

For small pipes sample rings may be cut from the pipe for measurement by calipers [37]. For large pipes permitting access, a beam micrometer may be used [9].

In all cases sufficient measurements of d should be taken to permit calculation of the standard deviation and thence an estimate of the uncertainty in diameter  $e_d$  to the required confidence level [28].

#### Measurement of Pipe Length L

Distance between pressure tappings may be determined accurately by steel tape for laboratory tests or by survey traverse for field tests [23]. Uncertainty in pipe length is usually insignificant compared to uncertainty in the other variables, except for buried pipes where construction records have been lost.

#### **Measurement of Pipe Elevation**

The elevation z of tapping points is required when the pressure head  $\frac{p}{\gamma}$  is being measured to locate the HGL (see Figure 1). For laboratory tests care is taken to lay the pipe straight and level using precise survey levelling, except where cases of joint displacement or deflection are being investigated [38].

For field tests survey levelling from adjacent bench marks is used [23]. Taking pressure readings at zero discharge to deduct the pressure head  $\frac{p}{\gamma}$  from a known static HGL has also been used [12], [23]. For very large pipes the surges produced by shutting a downstream control value to give a static HGL may persist for several hours [12].

When the HGL is detected by piezometer tubes, or differential manometer, the indicated head loss is independent of the pipe slope so no elevation data is required.

# **Measurement of Pressure**

The most common way of accessing the flow to measure pressure at a cross-section is to drill one or more holes through the wall of the pipe. The centre-lines of these wall tapping holes are normal to the pipe centre-line. For field tests on manufactured pipes a tapping band is required so usually only one hole per cross-section is used [9], [16], [23], [40]. For large tunnels and penstocks pressure tappings may be installed during construction specifically for pipe tests after commissioning [18], [42]. Vertical shafts used during construction have been used subsequently as piezometers for head loss tests [8], [34]. For laboratory tests one [24], [26], [29], [41], two [30], or four [15], [17] wall tapping holes per cross-section have been used.

When two or more tapping holes are used at a cross-section they are usually connected by a ring manifold to average the pressure, with a single offtake to the pressure measuring device [5], [6], [17]. To avoid air accumulation at an obvert tapping, or blocking by debris of an invert tapping, tapping holes should preferably be located in sectors of the cross-section between  $+\frac{\pi}{4}$  and  $-\frac{\pi}{4}$  rad. to the horizontal [5]. The angle between adjacent tapping holes should be constant.

Wall tappings overestimate the static pressure slightly, and the error increases as the diameter of the tapping hole increases [19]. The detailed specifications for wall tappings for constriction meters should also be applicable to pipe tests [27].

For manufactured pipes connected by flanged, socketed or sleeved joints the wall tappings are usually made a short distance upstream of the joint [17], [23], presumably to minimise flow interference from the joint upstream.

The number of tapping cross-sections which have been used to define the HGL for friction tests varies from two to twenty. Some laboratory tests have used a sufficient number to make an adequate statistical estimate of the energy gradient and its uncertainty [30], [38]. Many investigators, however, while careful in measuring pressure, discharge, etc, have used only two or three taps [15], [17], [26], [29], [36]. For field tests two or three tapping cross-sections are common, occasionally more are used [23], [34], [40].

Alternative means of accessing the flow for pressure measurement include annular slots [37] and static pressure probes [36], [38]. Static pressure probes may have an advantage over wall tappings for very rough pipes, where presence of a roughness element close to the hole may affect the pressure, or for brittle pipes where fracture of the wall on drill breakthrough produces a conical hole.

Devices used for measuring pressure include simple piezometric tubes [8], [17], [18], [30], [34], manometers and differential manometers [9], [18], [26], [29], [37], [38], Bourdon pressure gauges [16], [23], dead weight testers [6], [16], and pressure transducers [6], [41]. Distortion of pressure signals in plastic connecting tubes has been investigated by Carolus et al. [11].

# Measurement of Discharge

Laboratory tests have used volumetric tanks [17], [26], [38], constriction meters [15], pitot traverse [36], and electromagnetic flow meters [30]. Field tests have used orifice plates [23],

venturi meters [9], [34], [41], volumetric tanks [16], stream gauging [8], salt-velocity and colourprism methods [9], and correlation of discharge with turbine power output [6], [18].

When standard designs for constriction meters are used the calibration coefficients are given by the relative standard code.

#### **Measurement of Fluid Properties**

The fluid properties required for water flow are density  $\rho$  and dynamic viscosity  $\mu$ , both functions of water temperature. Unit weight  $\gamma = \rho g$  is required for finding pressure head  $\frac{p}{\gamma}$ . The kinematic viscosity  $\nu = \frac{\mu}{\rho}$  is required for use in friction factor equations.

Temperature read with a cheap mercury-in-glass thermometer has an uncertainty  $e_t$  of about 0.5°C. Temperature read by an industrial quality electronic thermometer has an  $e_t$  of about 0.1°C [22].

# **APPENDIX C: COMPUTER PROGRAM**

```
С
        PROGRAM PIPE.TEST
С
        FIND DIAMETER, ROUGHNESS K & MINOR LOSS COEFFICIENT FROM PIPE TESTS
С
        BY LOCATING MINIMUM LOG10 (SUM SQUARES RESIDUAL HEAD)
                 WITH GOLDEN SECTION SEARCH
С
С
        AUTHORS
С
           T.R. FIETZ AND K.B. HIGGS
С
           UNIVERSITY OF NEW SOUTH WALES, WATER RESEARCH LABORATORY
С
          AUGUST, 1989
С
С
        VARIABLES
С
С
        *=COMMON VARIABLE
С
          *ANU(NP)=KINEMATIC VISCOSITY (M^2/S) (FROM PIPE TESTS)
С
           ICASE=CASE NUMBER
С
                 =1, casen(1)='GIVEN DIA., FIND SAND GRAIN ROUGHNESS K'
С
                 =2, casen(2)='GIVEN DIA.& K, FIND MINOR LOSS COEFF'
С
                 =3, casen(3)='GIVEN DIA., FIND K & MINOR LOSS COEFF'
С
                 =4, casen(4) ='FIND DIA. & K'
С
                 =5, casen(5)='FIND DIA., K & MINOR LOSS COEFF'
С
         CASEN(5)=CASE DESCRIPTIONS
С
          D=PIPE DIA. (MM FOR I/O, M FOR CALCNS.)
С
          *DC=CURRENT DIA.(M)
С
          DG=DIA. AT GLOBAL MIN. (MM FOR I/O, M FOR CALCNS.)
C
          *DHIG=UPPER LIMIT OF DIA. FOR SEARCH (MM FOR I/O, M FOR CALCNS.)
*DINC=DIA. INCREMENT CONTAINING GLOBAL MIN. (MM FOR I/O, M FOR CALCNS.)
С
С
          *DLOW=LOWER LIMIT OF DIA. FOR SEARCH (MM FOR I/O, M FOR CALCNS.)
С
           FC=CURRENT DARCY F
С
           FNAME=SEQUENTIAL FILENAME FOR TEST DATA
С
С
              FILE INCLUDES JOB, L, NP, Q(NP), H(NP), ANU(NP)
С
          *G=GRAVITY
          *H(NP)=MEASURED HEAD LOSS (FRICTION + MINOR LOSS)(M) (FROM PIPE TESTS)
С
          *HN (NP) = ESTIMATED HEAD LOSS (M) (FROM GLOBAL SEARCH)
С
           HC=CURRENT CALCULATED HEAD LOSS (M)
С
          *JOB=PROJECT NAME
С
           K=SAND GRAIN ROUGHNESS (MM FOR I/O, M FOR CALCNS.)
С
          *KC=CURRENT K
С
           KG=K AT GLOBAL MIN. (MM FOR I/O, M FOR CALCNS.)
С
          *KHIG=UPPER LIMIT OF K FOR SEARCH (MM FOR I/O, M FOR CALCNS.)
С
          *KINC=K INCREMENT CONTAINING GLOBAL MIN. (MM FOR I/O, M FOR CALCNS.)
C,
          *KLOW=LOWER LIMIT OF K FOR SEARCH (MM FOR I/O, M FOR CALCNS.)
С
          *LEN=PIPE LENGTH(M)
С
          *MC=CURRENT MLC
С
           MG=MLC AT GLOBAL MIN.
С
          *MHIG=UPPER LIMIT OF MLC FOR SEARCH
С
          *MINC=MLC INCREMENT CONTAINING LOCAL MIN.
С
          *MLOW=LOWER LIMIT OF MLC FOR SEARCH
С
          *NP=NO. OBSERVATIONS FROM PIPE TESTS
С
          *NU=CURRENT KINEMATIC VISCOSITY (M^2/S)
С
          *PI=pi
С
          *Q(NP)=FLOWRATE(M^3/S)(FROM PIPE TESTS)
С
           QC=CURRENT Q(M^3/S)
С
           SUMSOS=LOG10 SUM SQUARES AT GLOBAL MIN.
С
С
         SUBROUTINE CALLS
С
С
         CASE 1 - GIVEN DIA., FIND SAND GRAIN ROUGHNESS K
С
                  KIP
С
                  AGOLD <- KSUMEH <- HLOSS <- YAOF
С
                 HLOSS <- YAOF
С
         CASE 2 - GIVEN DIA. & K, FIND MINOR LOSS COEFF
С
                 MTP
С
                  AGOLD <- MSUMEH <- HLOSS <- YAOF
С
                 HLOSS <- YAOF
С
         CASE 3 - GIVEN DIA., FIND K & MINOR LOSS COEFF
С
                 KIP
С
                 MIP
С
                 AGOLD <- MMID <- CGOLD <- MSUMEH <- HLOSS <- YAOF
С
                 HLOSS <- YAOF
С
        CASE 4 - FIND DIA. & K
С
```

DTP С KIP С AGOLD <- DMID <- CGOLD <- DSUMEH <- HLOSS <- YAOF С HLOSS <- YAOF С CASE 5 - FIND DIA., K & MINOR LOSS COEFF С DIP С KIP С С MIP AGOLD <- BIG <- BGOLD <- DMID <- CGOLD <- DSUMEH <- HLOSS <- YAM С HLOSS <- YAOF С С implicit real\*8 (a-h, o-z) common /set1/ anu(40),q(40),h(40),hn(40),np common /set2/ g,len,dc,kc,mc,nu,pi real\*8 len,k,kc,mc,nu dimension casen(5) character job\*200, casen\*40 common /set3/ dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc real\*8 dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc real\*8 kg,mg character fname\*15 external ksumeh, msumeh, mmid, dmid, big g = 9.8d0pi = 3.141593d0casen(1)='GIVEN DIA., FIND SAND GRAIN ROUGHNESS K ' casen(2)='GIVEN DIA. & K, FIND MINOR LOSS COEFF casen(3)='GIVEN DIA., FIND K & MINOR LOSS COEFF casen(4)='FIND DIA. & K casen(5)='FIND DIA., K & MINOR LOSS COEFF print \*, 'PIPE FLOW ANALYSIS PROGRAM' С if getarg is not available, delete next line and remove c from next read С call getarg(1,fname) print \* print \*, 'INPUT FILE NAME FOR TEST DATA:- ', fname read 1, fname С open (1, file=fname) rewind 1 read(1,1) job 1 format(a) read(1,\*) len read(1,\*) np if(np.gt.40) then print \*, 'WARNING - Program Dimensions Exceeded' print \*, 'Number of Data Values=', np,', Dimensions Set to 40' stop endif read(1,\*) ( q(i),h(i),anu(i), i=1,np ) close(1) С print \* print \*, job print print \*,' CASE 100 CASE! print \*, 'NUMBER DETAILS' write(6,2) (i,casen(i),i=1,5) 2 format(2x, i3, 4x, a40) print \* print \*, 'INPUT CASE NUMBER ' read \*, icase С if(icase.lt.1.or.icase.gt.5) then print \*,'Try again' goto 100 endif if(icase.eq.1) then print \*, 'INPUT PIPE DIA. IN MM. ' read \*,D D = D / 1000d0call kip (klow, khig, kinc)

```
Appendix C: Computer Program (Contd)
```

## Appendix C: Computer Program (Contd)

С

С

С

 $\mathbf{C}$ 

```
elseif(icase.eq.2) then
         print *, 'INPUT PIPE DIA. IN MM. '
         read *,D
         D = D / 1000d0
         print *, 'INPUT SAND GRAIN ROUGHNESS, IN MM. '
         read *,K
         K = K / 1000d0
         call mip (mlow, mhig, minc)
       elseif(icase.eq.3) then
         print *, 'INPUT PIPE DIA. IN MM. '
         read *,D
         D = D / 1000d0
         call kip (klow, khig, kinc)
         call mip (mlow, mhig, minc)
       elseif(icase.eq.4) then
         call dip (dlow, dhig, dinc)
         call kip (klow, khig, kinc)
       else
         call dip (dlow, dhig, dinc)
         call kip (klow, khig, kinc)
         call mip (mlow, mhig, minc)
       endif
       if (icase.eq.1) then
         dc = d
         mc = 0d0
         call agold(klow,khig,kinc,ksumeh, kg,sumsqs)
         dq = d
         mg = 0d0
       elseif(icase.eq.2) then
         dc = d
         kc = k
         call agold (mlow, mhig, minc, msumeh, mg, sumsqs)
         dg = d
         kg = k
       elseif(icase.eq.3) then
         dc = d
         call agold(klow, khig, kinc, mmid, kg, sumsqs)
         dg = d
         mg = mc
       elseif(icase.eq.4) then
         mc = 0d0
         call agold(klow,khig,kinc,dmid, kg,sumsqs)
         dq = dc
         mg = 0d0
       CASE 5
       else
         call agold (mlow, mhig, minc, big, mg, sumsqs)
         kg = kc
         dg = dc
       endif
        ********* FIND % ERROR IN ESTIMATED HYD. GRAD. ****************
       do 200 i = 1, np
         qc = q(i)
         nu = anu(i)
         call hloss(qc,dg,kg,mg, hc)
         hn(i) = hc
       continue
200
       print *
       print *
                           PROGRAM HED1'
       print *,'
                           ****** ****
       print *,'
       print *
       print *, 'JOB:-', job
       print *
       print *,'CASE TYPE:- ',icase,' ',casen(icase)
       print *
       print *
```

PIPE TEST RESULTS' print \*,' print \*,' \*\*\*\* \*\*\*\* \*\*\*\*\*\* if(icase.lt.4) write(6,3) d\*1000d0 =',f8.2,' mm') format ('INPUT PIPE DIAMETER 3 if(icase.eq.2) write(6,4) k\*1000d0 =',f8.4,' mm') format ('INPUT SAND GRAIN ROUGHNESS 4 print \* NU\*1E6' print \*,' HEAD Q print \*,' (L/S) (M^2/S)' (M) write(6,5) ( q(i)\*1000d0, h(i), anu(i)\*1e6 ,i=1,np) format(f9.2,2x,f8.4,2x,f8.2) 5 print \* print \* GLOBAL SEARCH DETAILS' print \*,' print \*,' \*\*\*\*\* \*\*\*\*\* \*\*\*\*\*\* if(icase.eq.4.or.icase.eq.5) write(6,6) dlow\*1000d0, dhig\*1000d0, dinc\*1000d0 format ('DIA. FROM ',f9.1,' MM, TO ',f9.1, 6 ' MM, IN INTERVAL ', f9.6,' MM') if(icase.ne.2) write(6,7) klow\*1000d0, khig\*1000d0, kinc\*1000d0 format('K FROM ',f9.3,' MM, TO ',f9.3, format('K FROM 7 ' MM, IN INTERVAL ', f9.6, ' MM') + if(icase.eq.2.or.icase.eq.3.or.icase.eq.5) write(6,8) mlow, mhig, minc +TO ', f9.3, 8 format ('MINOR LOSS COEFF. FROM', f9.3, ' + , IN INTERVAL ', f9.6) print \* print \* print \*,' GLOBAL MINIMUM' print \*,' \*\*\*\*\* \*\*\*\*\*\*\* if(icase.eq.4.or.icase.eq.5) write(6,9) dg\*1000d0 format ('ESTIMATED PIPE DIAMETER =',f10.2,' mm') 9 if(icase.ne.2) write(6,10) kg\*1000d0 format('ESTIMATED SAND GRAIN ROUGHNESS=',f10.4,' mm') 10 if(icase.eq.2.or.icase.eq.3.or.icase.eq.5) write(6,11) mg format ('ESTIMATED MINOR LOSS COEFF. =', f10.3) 11 write(6,12) 10.\*\*sumsqs format ('SUM SQUARES RESIDUAL H 12 =', F14.10)print \* print \* print \*,' ESTIMATED HEAD LOSS' print \*,' \*\*\*\*\*\*\* \*\*\*\*\*\*\*\* print \* print \*,' Q(L/S) HEAD (M) HEAD (M) PERCENT.' print \*,' FROM GLOBAL ERROR IN' FROM FROM TESTS print \*,' TESTS SEARCH HYD.GRAD.' do 300 i = 1, nppe = (hn(i) - h(i)) / h(i) \* 100d0write(6,13) q(i)\*1000, h(i), hn(i), pe 13 format(f9.2,2x,f9.4,3x,f9.4,3x,f8.3) 300 continue print \* print \* print \*,' END OF PROGRAM' end С SUBROUTINE DIP, INPUT DIA. RANGE & INCREMENT FOR SEARCH С С subroutine dip (dlow, dhig, dinc) real\*8 dlow, dhig, dinc print \* print \*, 'INPUT LOWER LIMIT OF DIA. FOR SEARCH, IN MM. ' read \*,dlow dlow = dlow / 1000d0

print \*, 'INPUT UPPER LIMIT OF DIA. FOR SEARCH, IN MM. '

Appendix C: Computer Program (Contd)

## Appendix C: Computer Program (Contd)

```
read *, dhig
      dhig = dhig / 1000d0
      print *, 'INPUT DIA. INCREMENT TO CONTAIN GLOBAL MIN., IN MM.,
      SUGGEST 1E-5'
      read *, dinc
      dinc = dinc / 1000d0
      end
      С
      SUBROUTINE KIP, INPUT K RANGE & INCREMENT FOR SEARCH
С
      С
      subroutine kip (klow, khig, kinc)
      real*8 klow,khig,kinc
      print '
      print *, 'INPUT LOWER LIMIT OF K VALUE FOR GLOBAL SEARCH, IN MM.'
      read *,klow
    klow = klow / 1000d0
      print *, 'INPUT UPPER LIMIT OF K VALUE FOR GLOBAL SEARCH, IN MM.'
      read *, khig
      khig = khig / 1000d0
      print *, 'INPUT K VALUE INCREMENT TO CONTAIN GLOBAL MINIMUM, IN MM., SUGGEST 1E-6'
      read *,kinc
      kinc = kinc / 1000d0
      end
      С
      SUBROUTINE MIP, INPUT MLC RANGE & INCREMENT FOR SEARCH
С
      С
      subroutine mip (mlow, mhig, minc)
      real*8 mlow,mhig,minc
      print *
      print *,
             'INPUT LOWER LIMIT OF MINOR LOSS COEFF. FOR SEARCH '
      read *,mlow
      print *, 'INPUT UPPER LIMIT OF MINOR LOSS COEFF. FOR SEARCH '
      read *, mhig
      print *, 'INPUT MINOR LOSS COEFF. INCREMENT TO CONTAIN GLOBAL MIN.,
      SUGGEST 1E-6'
    +
      read *, minc
 .
      end
      С
      SUBROUTINE KSUMEH, FIND LOG10 (SUM SQUARES RESIDUAL H)
С
      С
С
      subroutine ksumeh (k9, sum)
       K9=SAND GRAIN ROUGHNESS(M), INPUT
С
       SUM=LOG10(SUM SQUARES), RETURNED
С
       DC, MC, NP, NU, Q(*), H(*), ANU(*) COMMON
С
      implicit real*8 (a-h,o-z)
      common /set1/ anu(40),q(40),h(40),hn(40),np
common /set2/ g,len,dc,kc,mc,nu,pi
      real*8 k9, len, kc, mc, nu, m9
      sum = 0d0
      do 10 i = 1, np
        qc = q(i)
        hm = h(i)
        nu = anu(i)
        d9 = dc
        m9 = mc
        call hloss(qc,d9,k9,m9, hc)
        sum = sum + (hm - hc) ** 2
10
      continue
      sum = log10 (sum)
      end
      *****
С
      SUBROUTINE MSUMEH, FIND LOG10 (SUM SQUARES RESIDUAL H)
С
      MLC INPUT, DIA. & K HELD CONSTANT
С
      *****
С
```

## Appendix C: Computer Program (Contd)

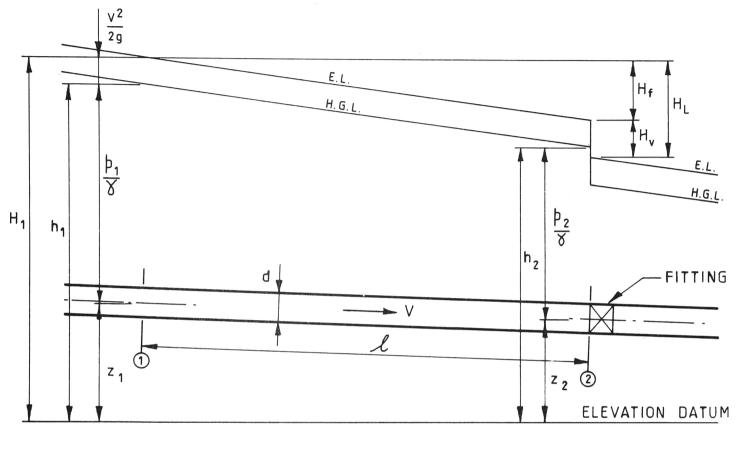
```
subroutine msumeh (m9, sum)
        M9=MINOR LOSS COEFF., INPUT
С
        SUM=LOG10 (SUM SQUARES), RETURNED
С
       DC, KC, NP, NU, Q(*), H(*), ANU(*) COMMON
С
       implicit real*8 (a-h, o-z)
       common /set1/ anu(40),q(40),h(40),hn(40),np
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 k9,len,kc,mc,nu,m9
       sum = 0d0
       do 10 i = 1, np
         qc = q(i)
         hm = h(i)
         nu = anu(i)
         k9 = kc
         d9 = dc
         call hloss(qc,d9,k9,m9, hc)
         sum = sum + (hm - hc) ** 2
10
       continue
       sum = log10 (sum)
       end
       С
С
       SUBROUTINE DSUMEH, FIND LOG10 (SUM SQUARES RESIDUAL H)
С
       DIA. INPUT, K & MLC HELD CONSTANT
       С
       subroutine dsumeh (d9, sum)
С
        D9=DIA.(M), INPUT
С
        SUM=LOG10 (SUM SQUARES), RETURNED
        HM=MEASURED HEAD LOSS (FROM PIPE TESTS)
С
С
        HC=CALCULATED HEAD LOSS (FROM COLEBROOK WHITE)
С
        KC, MC, NP, NU, Q(*), H(*), ANU(*) COMMON
       implicit real*8 (a-h,o-z)
       common /set1/ anu(40),q(40),h(40),hn(40),np
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 k9,len,kc,mc,nu,m9
       sum = 0d0
       do 10 i = 1, np
         qc = q(i)
         hm = h(i)
         nu = anu(i)
         k9 = kc
         m9 = mc
         call hloss(qc,d9,k9,m9, hc)
         sum = sum + (hm - hc) ** 2
10
       continue
       sum = log10 (sum)
       end
       С
       SUBROUTINE HLOSS, FIND HEAD LOSS FOR GIVEN Q, DIA., K, NU & MLC
С
       С
       subroutine hloss (qc,d9,k9,m9, hc)
        QC=FLOWRATE (M^3/S), INPUT
С
С
        D9=DIA.(M), INPUT
        K9=SAND GRAIN ROUGHNESS(M),
С
                                 INPUT
        M9=MINOR LOSS COEFF., INPUT
С
        HC=CALCULATED HEAD LOSS(M), RETURNED
С
С
        VC=VELOCITY (M/S)
С
        REYN=REYNOLDS NO.
        FC=CURRENT DARCY F
С
        G, PI, NU, LEN COMMON
С
       implicit real*8 (a-h,o-z)
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 k9,m9,len,kc,mc,nu,kd
       kd = k9 / d9
       reyn = 4d0 * qc / (pi * d9 * nu)
       if(reyn.lt.2100) then
         fc = 64d0 / reyn
       else
        call yaof(kd, reyn, fc)
```

```
Appendix C: Computer Program (Contd)
```

```
endif
       vc = 4d0 * qc / (pi * d9 * d9)
      hc = (fc * len / d9 + m9) * vc * vc / (2d0 * q)
       end
       С
С
       SUBROUTINE DMID, FIND LOG10 (SUM SQUARES RESIDUAL H)
       K INPUT, MLC HELD CONSTANT
C
С
       ***************
                           subroutine dmid (k9, f9)
       FIND LOCAL MIN.DIA FOR K9 INPUT, RETURNS LOG10 (SUM SQUARES RESIDUAL H)
С
С
       K9=SAND GRAIN ROUGHNESS(M), INPUT
С
       F9=LOG10 (SUM SQUARES), RETURNED
С
       DC, KC, MC, DLOW, DHIG, DINC COMMON
       implicit real*8 (a-h,o-z)
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 k9,len,kc,mc,nu
       common /set3/ dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc
       real*8 dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc
       external dsumeh
       kc = k9
       call cgold(dlow, dhig, dinc, dsumeh, dc, f9)
       end
       С
       SUBROUTINE BIG, FIND LOG10 (SUM SQUARES RESIDUAL H), MLC INPUT
С
       С
       subroutine big (m9, g9)
FIND LOCAL MIN. K, THEN LOCAL MIN. DIA. FOR M9 INPUT
С
        M9=MINOR LOSS COEFF., INPUT
С
        G9=LOG10 (SUM SQUARES RESIDUAL H), RETURNED
С
        DC, KC, MC, KLOW, KHIG, KINC COMMON
С
       implicit real*8 (a-h,o-z)
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 m9,len,kc,mc,nu
       common /set3/ dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc
       real*8 dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc
       external dmid
       mc = m9
       call bgold(klow,khig,kinc,dmid, kc,g9)
       end
       С
       SUBROUTINE MMID, FIND LOG10 (SUM SQUARES RESIDUAL H)
С
       K INPUT, DIA. HELD CONSTANT
С
       C
       subroutine mmid (k9, f9)
FIND LOCAL MIN. MLC FOR K9 INPUT
C
        K9=SAND GRAIN ROUGHNESS(M), INPUT
С
        F9=LOG10(SUM SQUARES RESIDUAL H), RETURNED
С
        DC, KC, MC, MLOW, MHIG, MINC COMMON
С
       implicit real*8 (a-h,o-z)
       common /set2/ g,len,dc,kc,mc,nu,pi
       real*8 k9,len,kc,mc,nu
       common /set3/ dlow,dhig,dinc,klow,khig,kinc,mlow,mhig,minc
       real*8 dlow, dhig, dinc, klow, khig, kinc, mlow, mhig, minc
       external msumeh
       kc = k9
       call cgold(mlow, mhig, minc, msumeh, mc, f9)
       end
       *****
С
       SUBROUTINE YAOF, FIND DARCY F, TURBULENT PIPE FLOW
С
       С
       subroutine yaof (kd, reyn, fc)
       SOLUTION OF COLEBROOK-WHITE EQUATION BY YAO'S METHOD
С
       REF. APPLE MANNZONE1, 5.10.85
С
         KD=PIPE RELATIVE ROUGHNESS, INPUT
С
         REYN=REYNOLDS NO., INPUT
С
        FC=DARCY F, RETURNED
C
```

```
Appendix C: Computer Program (Contd)
       implicit real*8 (a-h, o-z)
       real*8 kd
       z8 = kd / 3.7d0
       if(z8.le.0d0) then
         x7 = sqrt(0.02d0)
       else
         x7 = -1d0 / (2d0 \times \log 10 (z8))
       endif
       y7 = x7 * x7
100
       if(z8.lt.0d0) z8 = 0d0
       x8 = -1d0 / (2d0 \times \log 10 (z8 + 2.51d0 / (reyn \times x7)))
       fc = x8 * x8
       if( abs ((fc - y7) / y7).lt..0001d0) goto 200
       y7 = fc
       \mathbf{x}^{2} = \mathbf{x}^{8}
       goto 100
200
       return
       end
       С
       SUBROUTINE AGOLD, MIN. OF UNIMODAL FUNCTION BY GOLDEN SECTION SEARCH
С
        С
       subroutine agold (xlo, xhi, dx, funx, xmin, fxmin)
        REF. CHENEY & KINCAID, P.462
С
        XMIN AT FXMIN, MIN. PT. OF F(X), RETURNED
С
        XMIN IN RANGE XLO TO XHI
С
        DX=X INTERVAL CONTAINING XMIN
С
        XMIN TOLERANCE = + OR - DX/2
С
        FUNX=ENTRY NAME FOR SUBROUTINE GIVING F(X)
С
       implicit real*8 (a-h, o-z)
       external funx
       gsr = (sqrt (5.d0) - 1d0) * 0.5d0
       x0 = xlo
       x3 = xhi
       itnum = 1 + log (abs(dx / (x3 - x0))) / log (gsr)
x2 = x0 + gsr * (x3 - x0)
       call funx(x^2, y^2)
x1 = x0 + gsr * gsr * (x3 - x0)
        call funx(x1, y1)
        do 100 i = 1, itnum
          if(y2.gt.y1) then
         x3 = x2
         x^2 = x^1
         y^2 = y^1
         x1 = x0 + gsr * gsr * (x3 - x0)
         call funx(x1, y1)
         else
         x0 = x1
         x1 = x2
         y1 = y2
         x^2 = x^0 + gsr * (x^3 - x^0)
         call funx(x2, y2)
         endif
100
        continue
        xmin = (x1 + x2) * 0.5d0
        call funx(xmin, fxmin)
        end
        С
        SUBROUTINE BGOLD, MIN. OF UNIMODAL FUNCTION BY GOLDEN SECTION SEARCH
С
        IDENTICAL WITH AGOLD
С
        С
        subroutine bgold (xlo, xhi, dx, funx, xmin, fxmin)
        implicit real*8 (a-h,o-z)
       external funx
        gsr = (sqrt (5.d0) - 1d0) * 0.5d0
       x0 = xlo
        x3 = xhi
        itnum = 1 + log (abs(dx / (x3 - x0))) / log (gsr)
       x^2 = x^0 + gsr \star (x^3 - x^0)
```

```
Appendix C: Computer Program (Contd)
       call funx(x2, y2)
x1 = x0 + gsr * gsr * (x3 - x0)
call funx(x1, y1)
do 100 i = 1,itnum
          if(y2.gt.y1) then
          x3 = x2
          x^2 = x^1
          y2 = y1
          x1 = x0 + gsr * gsr * (x3 - x0)
          call funx(x1, y1)
          else
          x0 = x1
          x1 = x2
          y1 = y2
          x^2 = x^0 + gsr * (x^3 - x^0)
          call funx(x2, y2)
          endif
100
    ,
        continue
        xmin = (x1 + x2) * 0.5d0
        call funx(xmin, fxmin)
        end
        С
        SUBROUTINE CGOLD, MIN. OF UNIMODAL FUNCTION BY GOLDEN SECTION SEARCH
С
        IDENTICAL WITH AGOLD
С
        С
        subroutine cgold (xlo, xhi, dx, funx, xmin, fxmin)
        implicit real*8 (a-h,o-z)
        external funx
        gsr = (sqrt (5.d0) - 1d0) * 0.5d0
        x0 = x10
        x3 = xhi
        itnum = 1 + log (abs(dx / (x3 - x0))) / log (gsr)
x2 = x0 + gsr * (x3 - x0)
        call funx(x^2, y^2)
x1 = x0 + gsr * gsr * (x3 - x0)
        call funx(x1, y1)
        do 100 i = 1, itnum
          if(y2.gt.y1) then
          x3 = x2
          x2 = x1
          y^2 = y^1
          x1 = x0' + qsr * qsr * (x3 - x0)
          call funx(x1, y1)
          else
          x0 = x1
          x1 = x2
          y1 = y2
          x^2 = x^0 + gsr * (x^3 - x^0)
          call funx(x^2, y^2)
          endif
100
        continue
        xmin = (x1 + x2) * 0.5d0
        call funx(xmin, fxmin)
        end
```



## FIGURE 1 HEAD LOSS NOTATION

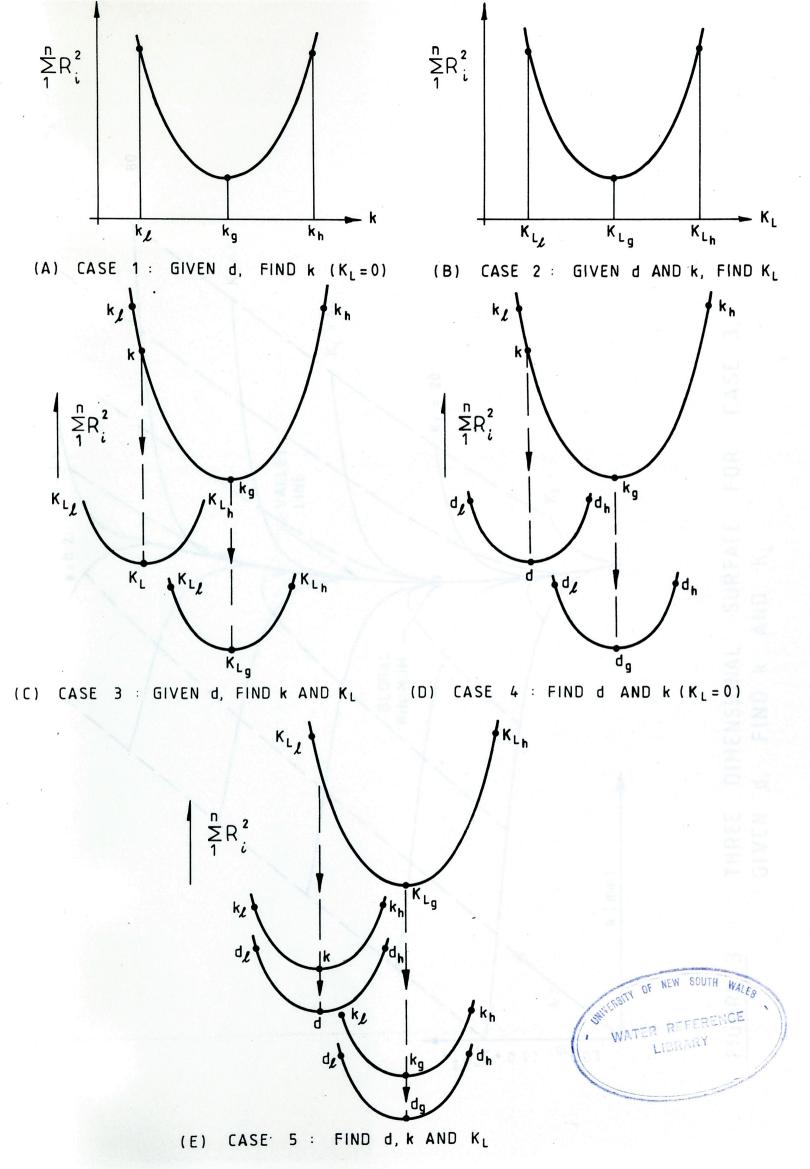


FIGURE 2 FIVE CASES OF PIPE TEST ANALYSIS

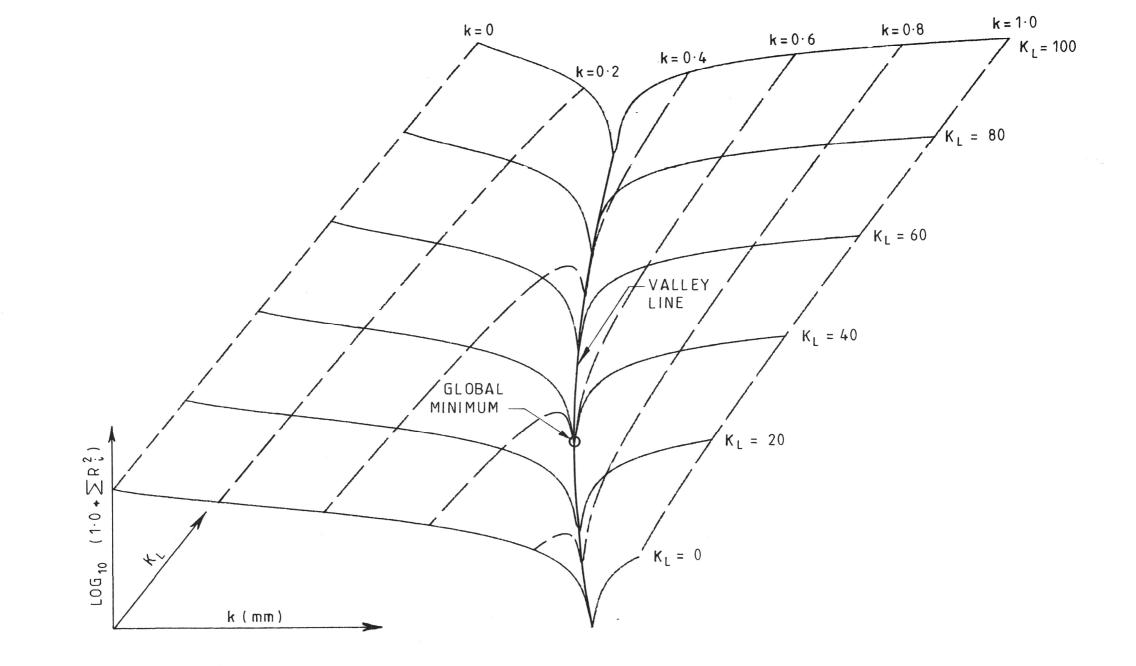
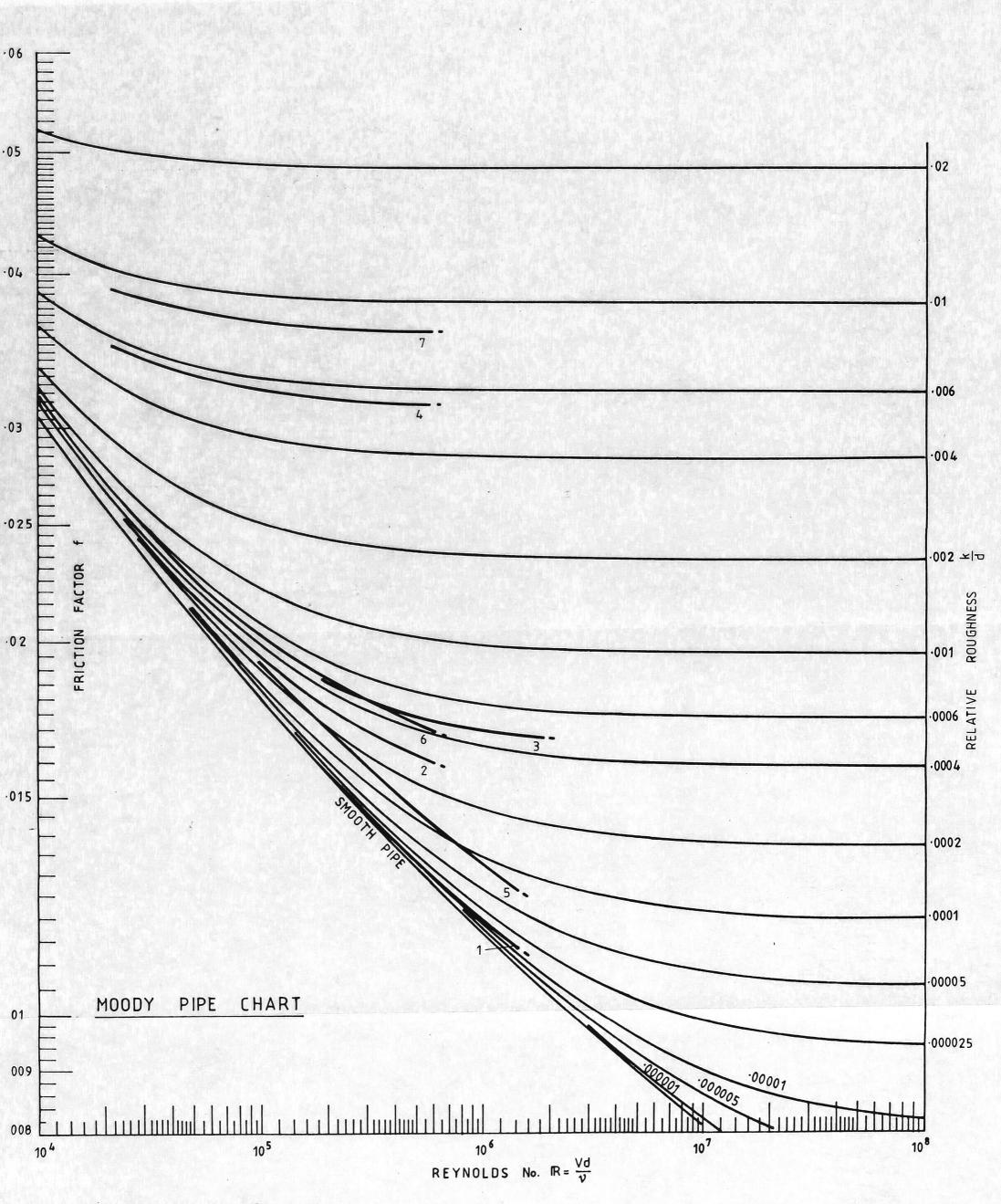
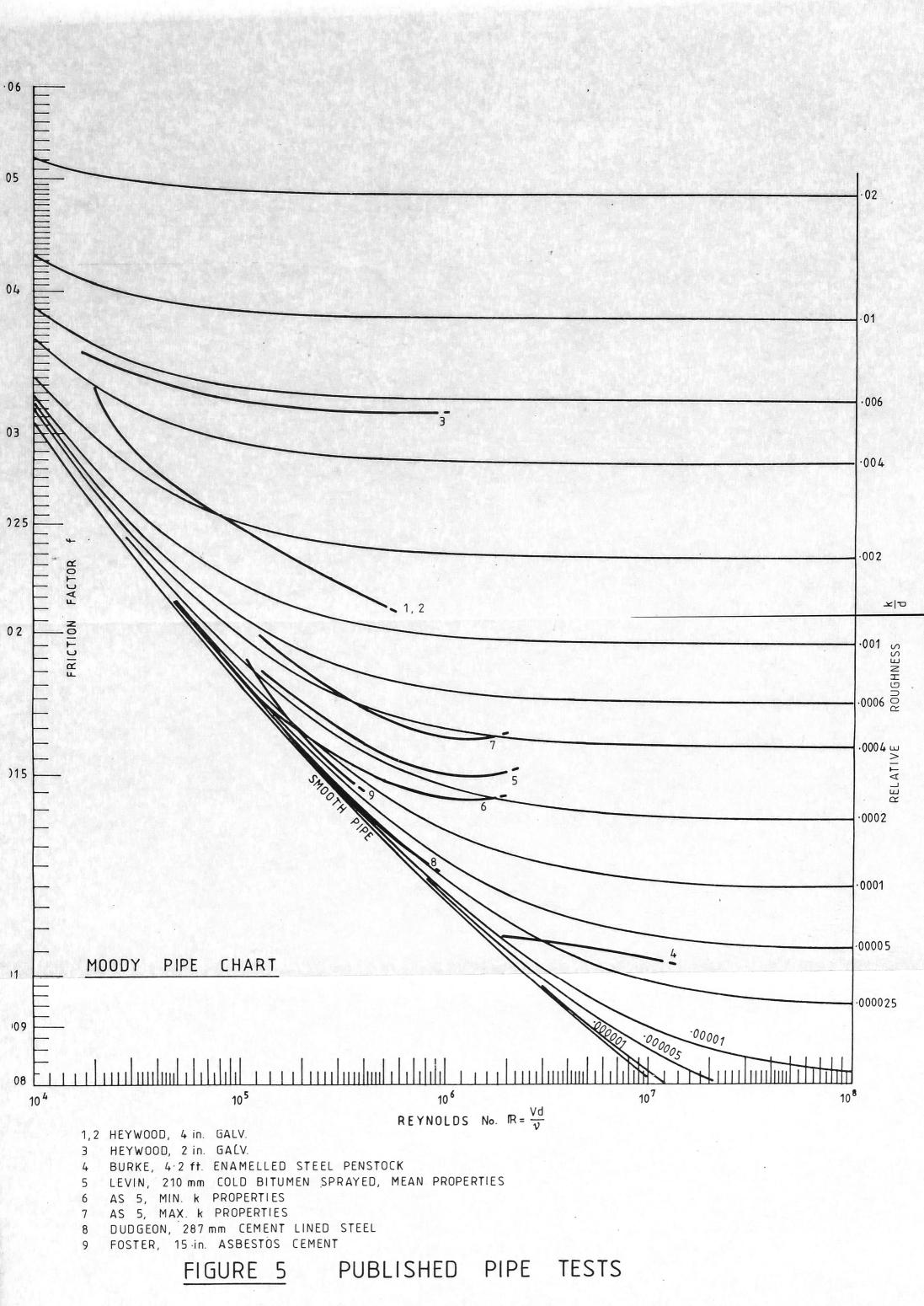
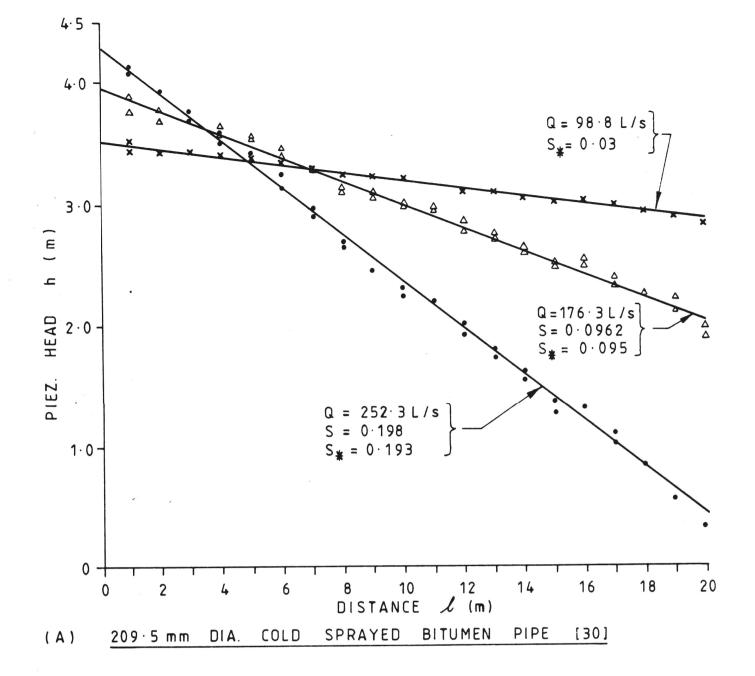


FIGURE 3 THREE DIMENSIONAL SURFACE FOR CASE 3, GIVEN d, FIND k AND KL



- 1 POLYETHYLENE, k = 0.003 mm  $K_{L} = 0$
- 2 CONCRETE, GOOD, k = 0.03 mm  $K_{L} = 0$
- 3 CI UNCOATED, GOOD, k = 0.15 mm  $K_{L} = 0$
- 4 CI UNCOATED, POOR, k = 0.6 mm  $K_{L} = 0$
- 5 POLYETHYLENE, k = 0.003 mm  $K_L = 5$
- 6 CONCRETE, GOOD, k = 0.03 mm  $K_L = 10$
- 7 CI UNCOATED, POOR, k = 0.6 mm  $K_{L} = 40$
- FIGURE 4 SYNTHETIC PIPE TESTS





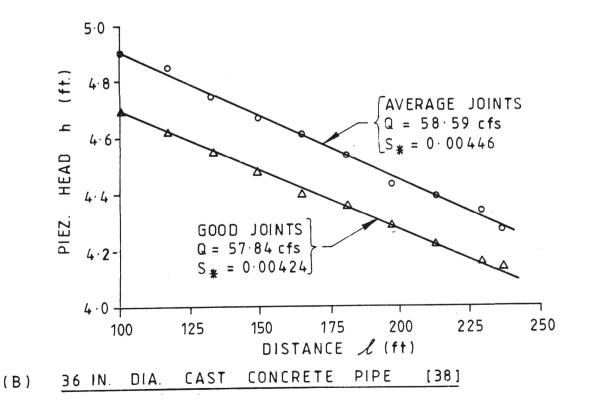
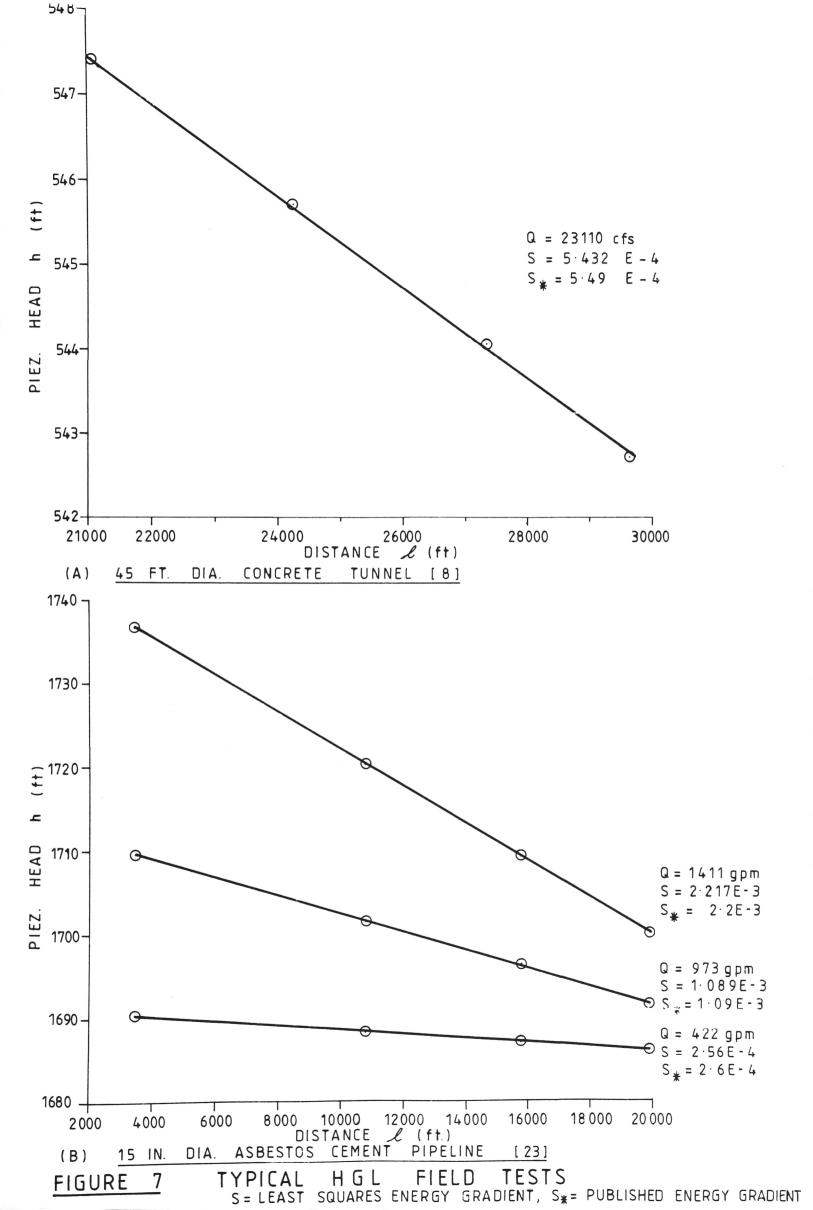
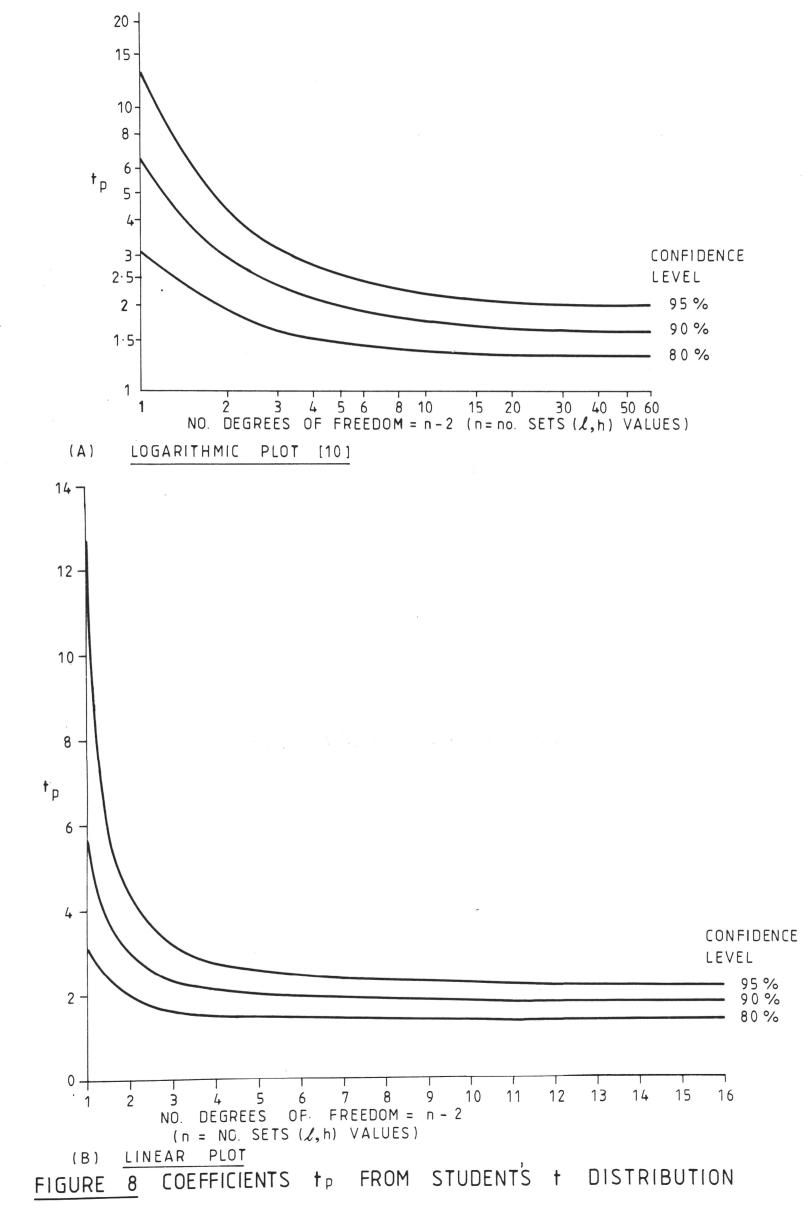


FIGURE 6

TYPICAL HGL, LABORATORY TESTS, S = LEAST SQUARES ENERGY GRADIENTS S\* = PUBLISHED ENERGY GRADIENT





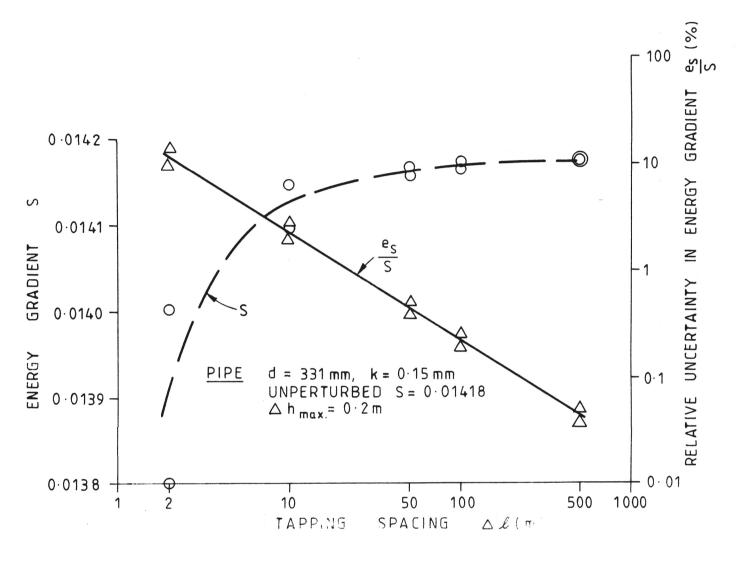
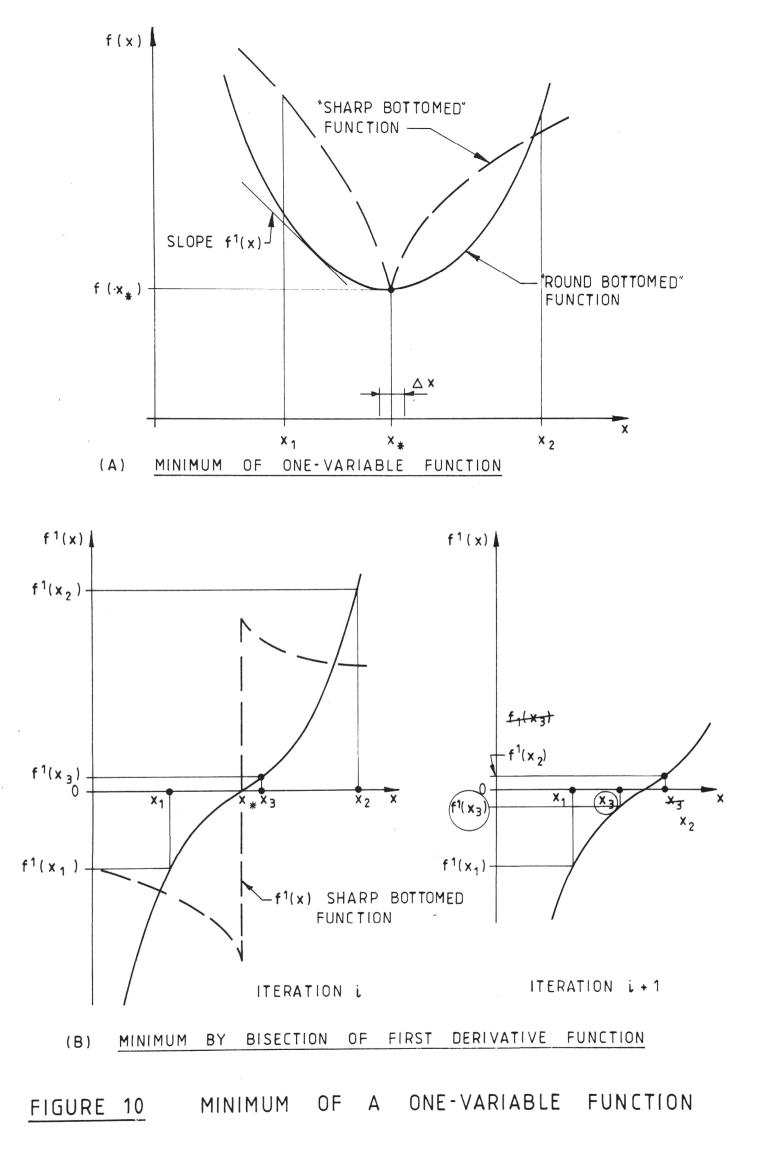
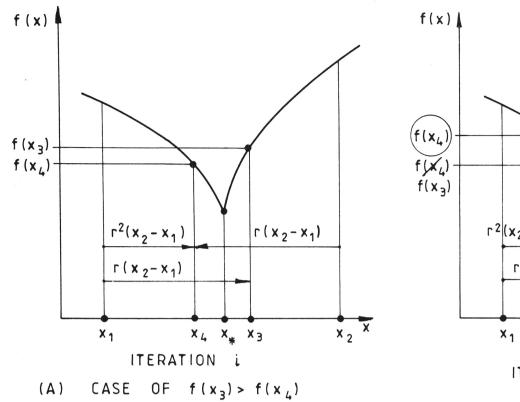
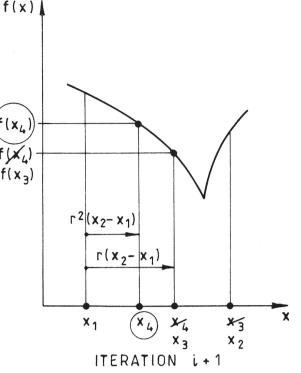
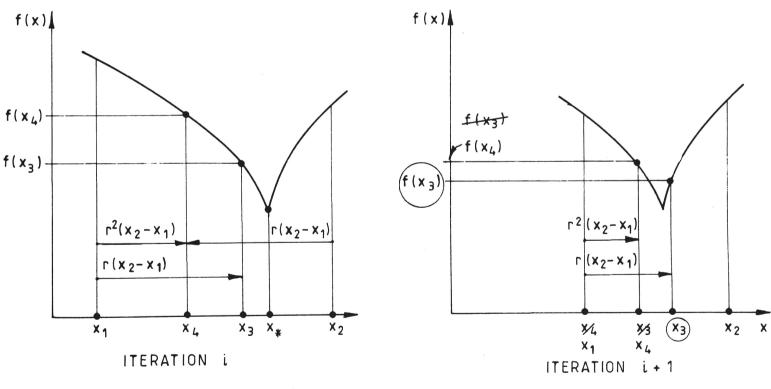


FIGURE 9 EFFECT OF TAPPING SPACING









(B) CASE OF  $f(x_3) \leq f(x_4)$ 

FIGURE 11 MINIMISATION OF A ONE-VARIABLE FUNCTION BY GOLDEN SECTION SEARCH